



Advanced Aeroelasticity. Máster Universitario en Ingeniería Aeronáutica

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# Aeroelastic and Dynamic Structural Analysis of a non-tapered wing

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August 23, 2019



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# Chapter 1

## Introduction

Structural Dynamics is a field of major importance in the design of every engineering system to deeply understand its behaviour subjected to dynamic loading. And when it comes to the concept design of aerospace structures, studying how these dynamic forces will influence the performance of the vehicle is of great interest as well.

Deeply related to the latter is the second engineering field that must be address during the early stages of aircraft design, and that is Aeroelasticity. Aeroelastic phenomena can have a significant influence on the design of flight vehicles. Indeed, these effects can greatly alter the design requirements that are specified for the disciplines of performance, structural loads, flight stability and control, and even propulsion. In addition, aeroelastic phenomena can introduce catastrophic instabilities of the structure that are unique to aeroelastic interactions and can limit the flight envelope.

The interaction between a lifting surface and the fluid field can eventually lead under certain conditions to the so-called Flutter phenomenon. Flutter is a dynamic instability, and it can be regarded as a response to a harmonic auto-excited problem with divergent oscillations in which aerodynamic forces couple with the normal modes of the structure, as a result of the interaction of elastic, inertial and aerodynamic forces.

In this document a thorough analysis of flutter will be presented, applied on a wing box which has been discretized using a FEM tool, MSC. NASTRAN, to cover both the dynamic structural analysis as well as the aeroelastic solution to flutter. Once the structural part has been detailed, a discussion about the flutter speed obtained with the Nastran SOL 145 will be presented, followed by some other solutions.

### 1.1 Structural model

The structure analysed is a typical aircraft wing box that has the following specifications:

- Length: 6.1 *m*
- Width: 1.22 *m*
- Mass: 3283 *Kg*

On the outtermost rib a pod has been installed:

- Length: 3.05 *m*
- Mass: 328 *Kg*
- Inertia: 161.7 *Kgm*<sup>2</sup>

Proceeding to the FEM model, the latter has been modelled as follows:

- 66 grids.
- 60 rods.
- 20 surface elements.
- Some other rigid bars to model a pod where a missile will be later installed.

These elements can be seen in Figure 1.1:

## 1.2 FEM Model

- The model contains the following elements:
- GRID: it is used to define nodes in the space
- CONM2: used to define punctual masses and inertia along with their properties
- CROD: rod elements with tension, compression, and torsional capabilities.
- CQUAD4: surface element that links four nodes.
- RBAR: rigid bar element that links two nodes, being the displacement of one node dependent from the other one. RBARS elements cannot be deformed.
- RBE3: rigid element that links nodes with independent degrees of freedom and nodes with dependent degrees of freedom. It is used in order to transmit loads and mass.



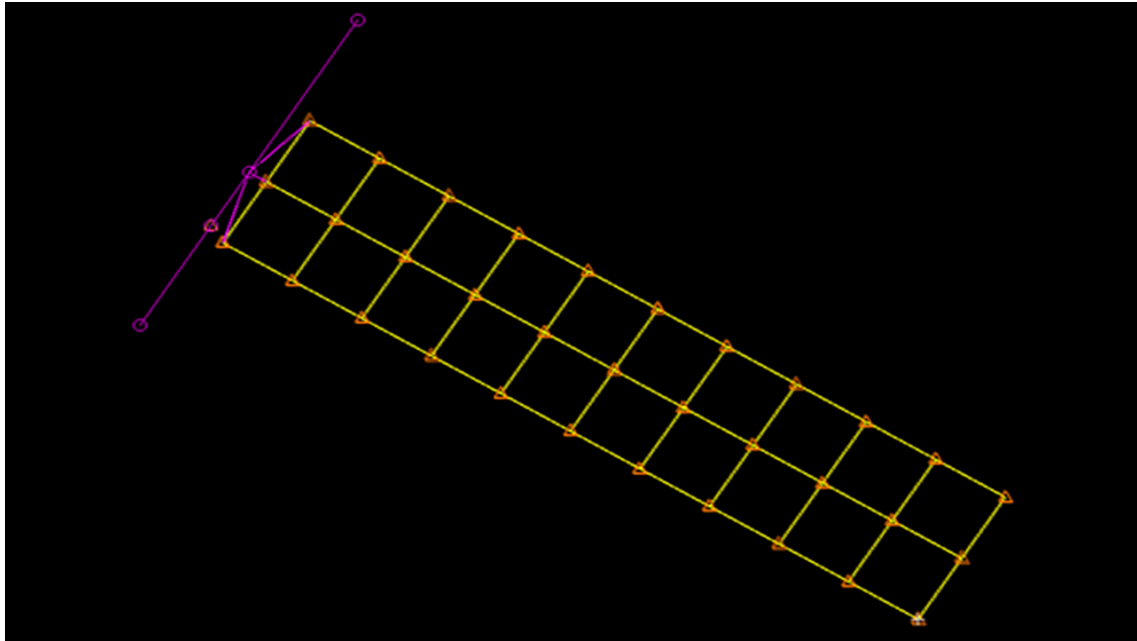


Figure 1.1: FEM Model of the wing box

- SPC: single point constraint. It is used to set boundary conditions in terms of displacements.

The reference frame used for the construction of the model has the y-axis along the wing span and the x-axis normal to it. The z-axis is perpendicular to the other two in order to obtain a right-handed triad.



# Chapter 2

## Normal Modes

Once the geometric and FEM model has been created, the second step was to edit the Nastran *.bdf* file to configure the solution and obtain the normal modes of vibration. For the latter, the SOL 103 command has been used.

Firstly, the wing box was clamped at the root, editing the bdf file to constrain the corresponding nodes. Secondly, the rest of the SOL 103 parameters were set to obtain the natural modes of free vibration below 50 Hz.

The equation of the system, imposing free vibration and zero damping:

$$[M]\{\ddot{u}\} + [K]\{u\} = 0 \quad (2.1)$$

And considering harmonic motion:

$$\{u\} = \{\phi\}e^{i\omega t} \quad (2.2)$$

$$([K] - \omega^2[M])\{\phi\} = 0 \quad (2.3)$$

Finding the non-trivial roots of this determinant poses an eigenvalue problem, where  $\lambda = \omega^2$ . Nastran solves it finding the eigenvalues to calculate the normal modes.

To extract these eigenvalues the Lanczos method has been set as parameter of the SOL 103 configuration. The fringe of each mode obtained by running the solution is presented in the next section, along with its natural frequency.

## 2.1 Fringe representation of normal modes

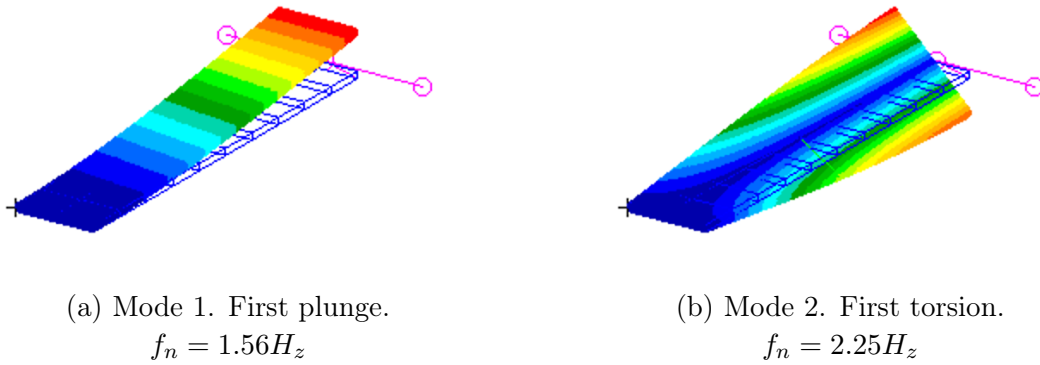


Figure 2.1: Modes 1 and 2

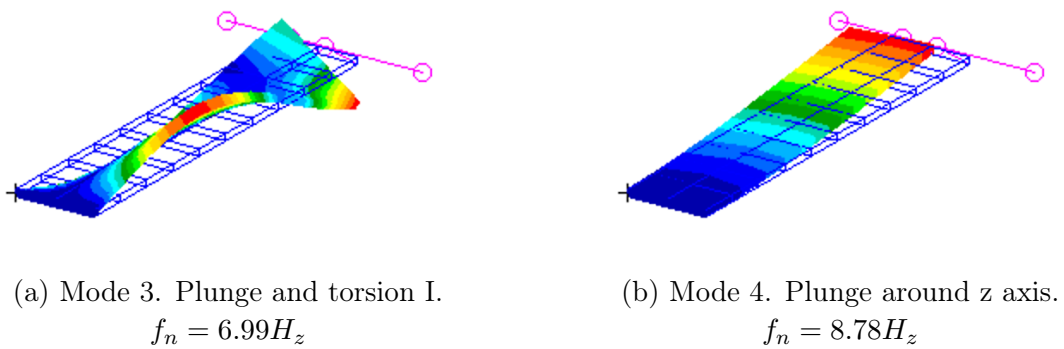


Figure 2.2: Modes 3 and 4

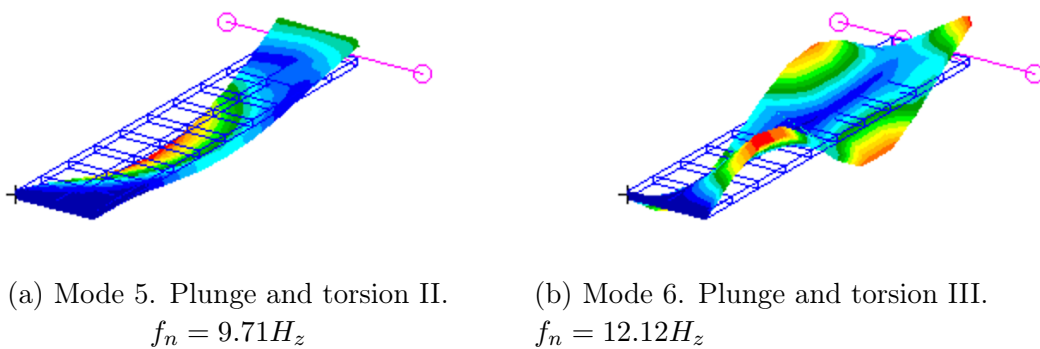
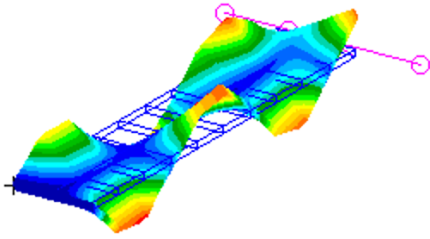
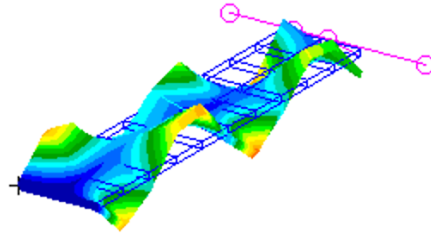


Figure 2.3: Modes 5 and 6

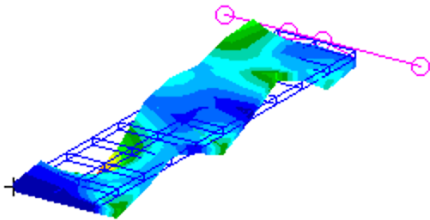


(a) Mode 7. Plunge and torsion IV.  
 $f_n = 17.08H_z$

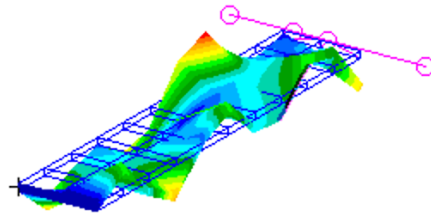


(b) Mode 8. Plunge and torsion V.  
 $f_n = 21.18H_z$

Figure 2.4: Modes 7 and 8

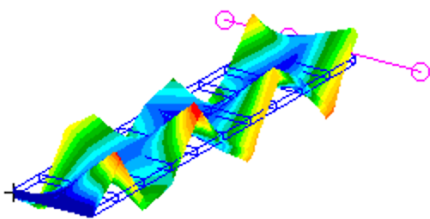


(a) Mode 9. Plunge and torsion VI.  
 $f_n = 24.62H_z$

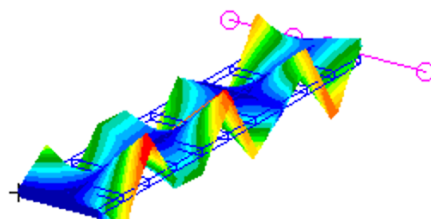


(b) Mode 10. Plunge and torsion VII.  
 $f_n = 25.67H_z$

Figure 2.5: Modes 9 and 10

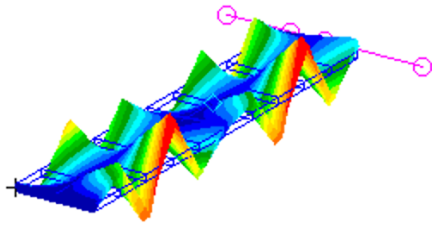


(a) Mode 11. Plunge and torsion VIII.  
 $f_n = 28.62H_z$

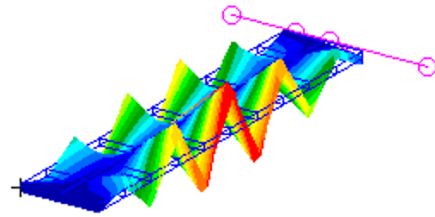


(b) Mode 12. Plunge and torsion IX.  
 $f_n = 31H_z$

Figure 2.6: Modes 11 and 12

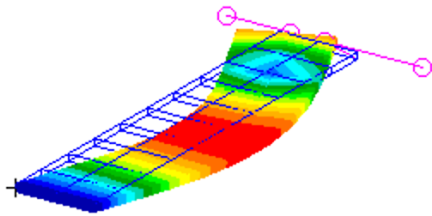


(a) Mode 13. Plunge and torsion X.  
 $f_n = 32.8H_z$

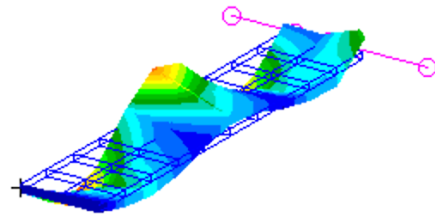


(b) Mode 14. Plunge and torsion XI.  
 $f_n = 33.87H_z$

Figure 2.7: Modes 13 and 14



(a) Mode 15. Plunge and torsion XII.  
 $f_n = 38.25H_z$



(b) Mode 16. Plunge and torsion XIII.  
 $f_n = 41.98H_z$

Figure 2.8: Modes 15 and 16

# Chapter 3

## Flutter Equation

It is of great importance to know which are the flight conditions under which the structure suffers instabilities such as divergence or flutter.

In order to find them, a dynamic analysis must be carried out. The equation of a dynamic system with damping can be expressed as follows:

$$[M_{aa}]\{\ddot{u}_a(t)\} + [B_{aa}]\{\dot{u}_a(t)\} + [K_{aa}]\{u_a(t)\} = \{P_a(t)\} \quad (3.1)$$

Where  $u_a$  is the vector of nodal displacements of the a-set,  $M_{aa}$  is the a-set mass matrix,  $B_{aa}$  is the a-set damping matrix,  $K_{aa}$  is the a-set stiffness matrix and  $P_a$  is the a-set vector of nodal aerodynamic forces.

$P_a$  can be expressed as a function of the nodal displacements as well:

$$\{P_a(t)\} = q_\infty \int_0^t \left[ H_{aa} \left( \frac{2U_\infty}{c}(t - \tau), M_\infty \right) \right] \{u_a(\tau)\} d\tau, \quad (3.2)$$

where  $q_\infty$  is the dynamic pressure ( $q_\infty = \frac{1}{2}\rho_\infty U_\infty^2$ ) and  $H_{aa}$  is the step response of the system for each nodal displacement.

In order to simplify the analysis, the nodal displacements will be considered to be a combination of the modes of free vibration  $\phi_{ah}$ .

$$\{u_a(t)\} = [\phi_{ah}]\{u_h(t)\} \quad (3.3)$$

Where  $u_h(t)$  are the modal displacements or modal coordinates.

The modes used are the ones below 50 Hz obtained in the previous section. It is considered that those modes approximate well enough the nodal displacements, as flutter is a low frequency phenomenon.

The dynamic equation multiplied by  $\phi_{ah}^T$ :

$$[\phi_{ah}^T][M_{aa}][\phi_{ah}]\{\ddot{u}_h\} + [\phi_{ah}^T][B_{aa}][\phi_{ah}]\{\dot{u}_h\} + [\phi_{ah}^T][K_{aa}][\phi_{ah}]\{u_h\} = q_\infty \int_0^t [\phi_{ah}^T] \left[ H_{aa} \left( \frac{2U_\infty}{c}(t-\tau), M_\infty \right) \right] [\phi_{ah}]\{u_h(\tau)\} d\tau \quad (3.4)$$

Or expressing it in terms of the generalized stiffness, mass and aerodynamic response matrices:

$$[M_{hh}]\{\ddot{u}_h(t)\} + [B_{hh}]\{\dot{u}_h(t)\} + [K_{hh}]\{u_h(t)\} = q_\infty \int_0^t \left[ Q_{hh} \left( \frac{2U_\infty}{c}(t-\tau), M_\infty \right) \right] \{u_h(\tau)\} d\tau \quad (3.5)$$

There are different methods to solve these equations. Some of them allow to obtain more accurate results than others but at higher computational cost. A quick method is the so-called K-method or V-g method.

### 3.1 K Method

The main idea of this method is that flutter occurs when one mode reaches simple harmonic motion, i.e. when the real part of the eigenvalue is 0 (zero damping), while the rest of the modes are still convergent (damped modes). Keeping in mind this idea, a way to find the boundary between stable and unstable (the flutter boundary) behaviour is by assuming harmonic motion.

$$u_h(t) = \hat{u}_h e^{i\omega t} \quad (3.6)$$

Another fundamental reason to do this is that non-steady aerodynamic forces are very difficult to calculate. In order to avoid this difficulty, recalling the already mentioned idea about harmonic motion and flutter, it can be assumed that aerodynamic forces are also harmonic, which are well known. To study the system motion in the frequency domain a Laplace transformation is applied to equation 3.5:

$$\left( -\omega^2[M_{hh}] + i\omega[B_{hh}] + [K_{hh}] - q_\infty \left[ Q_{hh} \left( \frac{2U_\infty}{c}\omega, M_\infty \right) \right] \right) \{u_h(\omega)\} = 0 \quad (3.7)$$



Where  $\{u_h(\omega)\}$  and  $[Q_{hh}(\frac{2U_\infty}{c}\omega, M_\infty)]$  are the Laplace transforms of  $\{u_h(t)\}$  and  $[Q_{hh}(\frac{2U_\infty}{c}t, M_\infty)]$ .

A few comments are remarkable in order to explain how damping is modeled:

- It is assumed that the model has no damping, although it is known that it has a structural damping of 0,03.
- The K method assumes an artificial damping ( $g$ ) in the sense that it is not a physical damping present in the structure, but the damping that the structure would require in order to have harmonic motion as it was assumed before.
- When this artificial damping it is calculated, it is compared with the real one. If the artificial damping is bigger than the real one, that means that the damping required to force harmonic motion it is higher than the damping that the structure has, so the motion will be unstable. On the other hand, if the artificial damping is lower than the real one, that means that the structure has enough damping to make the motion convergent.

$$[B_{hh}] = \frac{g_{hh}}{\omega} [K_{hh}] \quad (3.8)$$

Introducing the reduced frequency  $k = \frac{c\omega}{2U_\infty}$ , where  $c$  is the wing chord:

$$\left( \left( \frac{2kU_\infty}{c} \right)^2 [M_{hh}] - (ig_{hh} + 1)[K_{hh}] + \frac{1}{2}\rho_\infty U_\infty^2 [Q_{hh}(k, M_\infty)] \right) \{u_h\} = 0 \quad (3.9)$$

As the trivial solution has no interest, the flutter solution is the one that makes the determinant equal to zero:

$$\det \left( \left( \frac{2kU_\infty}{c} \right)^2 [M_{hh}] - (ig_{hh} + 1)[K_{hh}] + \frac{1}{2}\rho_\infty U_\infty^2 [Q_{hh}(k, M_\infty)] \right) = 0 \quad (3.10)$$

The latter is the flutter equation of the wing. As it is a complex equation, the reduced frequency of each mode ( $k$ ) and the structural damping required to have harmonic motion for each mode ( $g$ ) are obtained as a function of the flight conditions. If the  $g$  obtained is greater than the actual structural damping ( $g_{structure}$ ), the response will be unstable. If it is lower, it will be stable. Therefore the intersections between the (V- $g$ ) curve and  $g = g_{structure}$  shows where the system becomes unstable. Their corresponding flight conditions are the ones associated to the beginning of flutter and must be avoided during flight.



lated running the Nastran SOL 145 algorithm, which for this case Doublet-Lattice theory has been applied on every panel section over the wing. The parameter configuration has been described in section 9.1.

## **3.2 Static instability: Divergence**

After obtaining the flutter equation (3.10) the divergence equation can be easily obtained. It is only necessary to make  $k = 0$ , as the static case has a null frequency. Later on, the V-g method results will be shown and this instability will be identified as it is the one with  $k = 0$ .



# Chapter 4

## PK Matched Diagram

In order to obtain the flutter boundary, that is, the velocity at which the phenomenon of flutter starts, several mathematical methods can be used. For the present report, the PK method has been applied. It combines the idea of two methods:

- The P method: It solves the equation of flutter directly with non-steady aerodynamics theories and the unknowns are the real and imaginary parts of the eigenvalues. It is known from literature to be the most accurate procedure, but it has the disadvantage that it is not always possible to calculate the unsteady aerodynamic forces.
- The K method: Explained in the previous section. Harmonic motion and harmonic aerodynamic forces are assumed as a simplification.

In 1971, Hassig demonstrated that the approximation of harmonic motion and harmonic aerodynamics forces used in the K method was inadequate in some cases in which the predictions were wrong. However, the PK method solved this problem relaxing the hypothesis of harmonic motion, assuming only harmonic aerodynamics forces.

The free-stream air speed  $U_\infty$  is obtained as a function of the free-stream density  $\rho_\infty$  and Mach number  $M_\infty$ , according to the International Standard Atmosphere. Therefore, the result obtained is a PK-Matched Diagram.

Nastran has been used to carry out the flutter analysis by using the PKNL method. The studied case is a cruise flight at  $M_\infty = 0.8$  and altitude within the range  $[0\text{km} < h < 32\text{km}]$ . Further details can be read in the appendix where the .bdf file is included for Nastran analysis.

The following diagrams have been obtained:

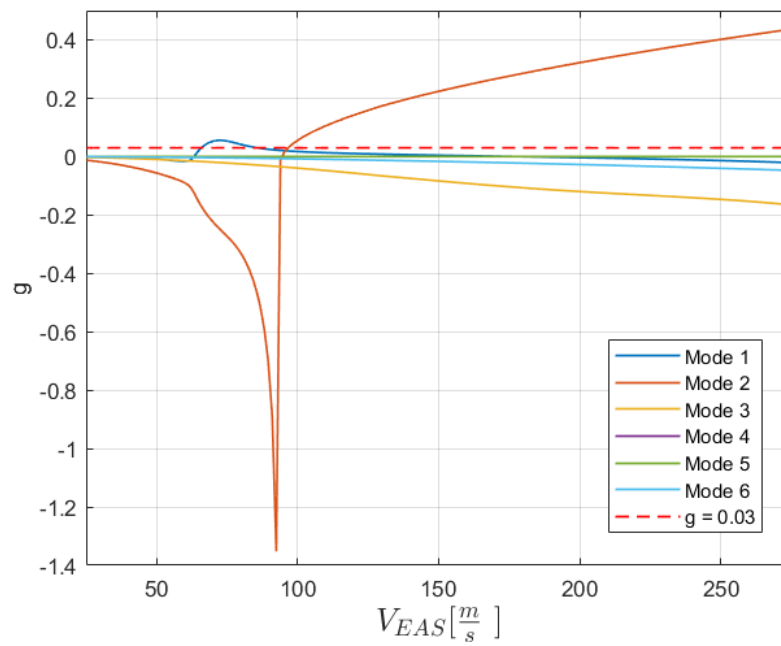


Figure 4.1: V-g diagram

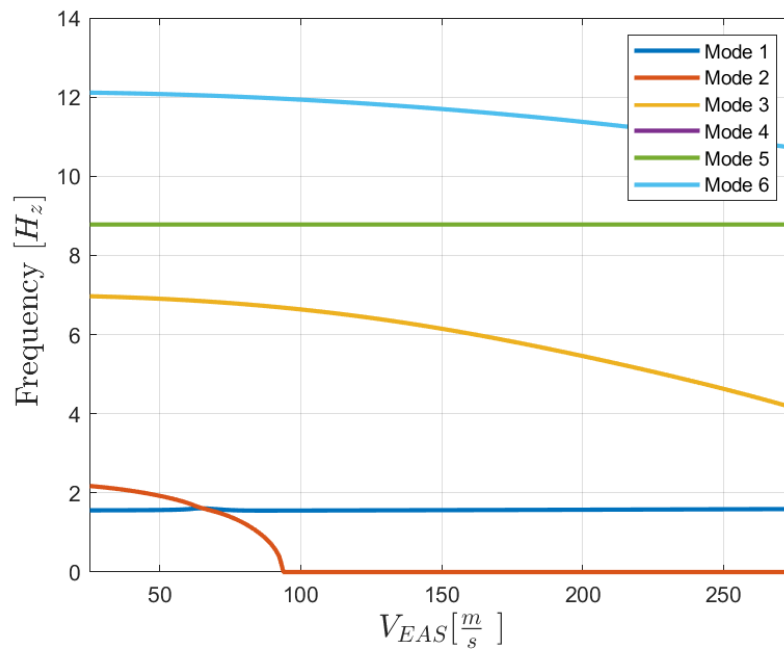


Figure 4.2: V-f diagram

Figure 4.1 shows the relation between  $g$  and  $V_{EAS}$  and figure 4.2 shows the relation between harmonic oscillation frequency and  $V_{EAS}$  for the first 6 modes of vibration.

## 4.1 Flutter

The parameter  $g$  can be considered as an artificially structural damping. As mentioned previously, it is the required value of structural damping to have harmonic motion. The flutter boundary begins at  $V_{EAS}|_{flutter} = 65.8 \frac{m}{s}$ , where the damping of the first mode is equal to 0.03 (see figure 4.1), which is the real structural damping ( $g_{structure}$ ). Sometimes, flutter is described as a phenomenon of energy transfer between modes [2], which starts being critical when their two frequencies coalesce. This behaviour is shown in the V-f diagram (4.2), where the frequency of the second mode approaches the first one until they are equal for  $V_{EAS} \simeq V_{EAS}|_{flutter}$ . It is remarkable to say that a first step in order to avoid flutter is to separate the frequencies of the modes that coalesce.

The flutter boundary obtained can be physically regarded as a complete dynamic interaction between the wing structure and airflow. For any value of speed less than  $V_{EAS}|_{flutter} = 65.8 \frac{m}{s}$ , any disturbance of the wing gets damped with exponentially decreasing amplitudes. It could be said that air provides the required damping to attenuate the disturbance. Above the flutter speed, however, the air provides the sufficient negative damping, and instead of decreasing the oscillatory motion created by the disturbance, the amplitude starts increasing exponentially.

In some cases, increasing speed after  $V_{EAS}|_{flutter}$  will continuously increase  $g$  and the system will diverge faster. With this wing, if  $V_{EAS}$  is high enough,  $g$  decreases and gets into the stable region again (below the red dashed line). This event happens at  $V_{EAS} = 85.8 \frac{m}{s}$ .

## 4.2 Divergence

The V-g curve for the second mode shows a singularity at  $V_{EAS} = 93.9 \frac{m}{s}$ , where  $g$  goes from having a big margin of stability to the unstable region. At the same  $V_{EAS}$ , its frequency becomes equal to zero (see 4.2). Therefore this instability is considered to be a static divergence.





# Chapter 5

## Joining Wing and Pod with Springs.

Initially, the junction between wing and pod has been considered to be rigid. The previous results have been obtained with that configuration. In this chapter, the pod has been joined to the external wing rib by installing springs in the degrees of freedoms of vertical displacement (along z-axis) and torsion (around y-axis). The rest of degrees of freedom will be joined in a rigid way. From now on, the value of the stiffness of each spring will be called  $k_v$  for the longitudinal one, and  $k_\theta$  for the torsional one. Both of them are ideal springs with natural elongation equal to zero. Figure 5.1 shows the joined grids with springs. Grid 20005 is part of the external wing rib and Grid 20001 belongs to the pod.

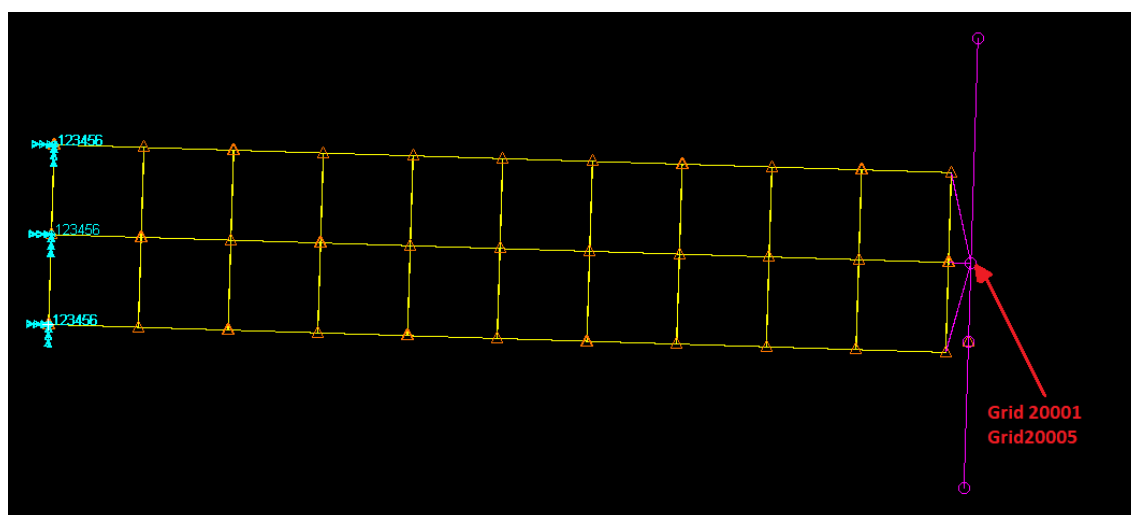


Figure 5.1: Wing-Pod Joint

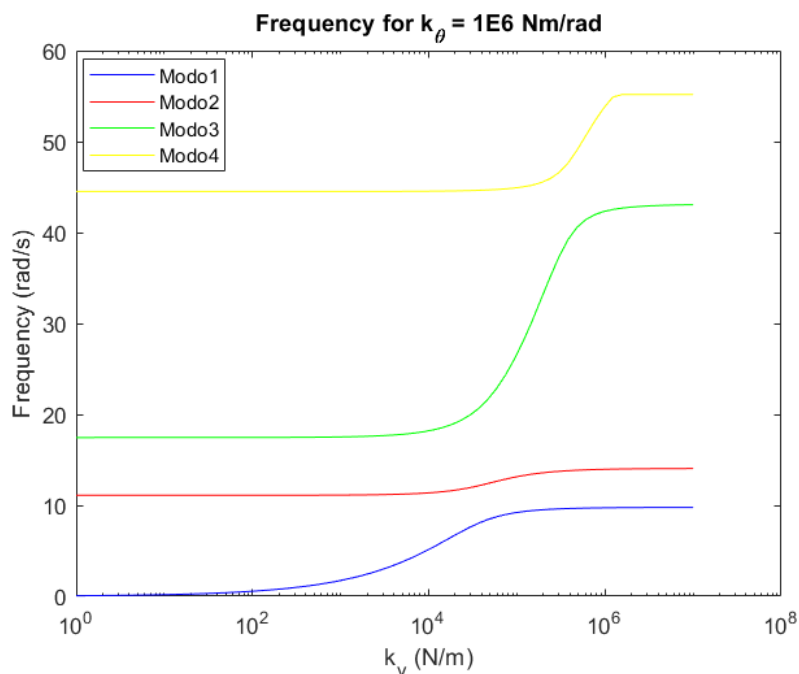
The Bulk Data Entry of Nastran used to model the connections between both modes are CELAS2 for the degrees of freedom with springs, and MPC for the rigid connections (see Fig. 5.2).

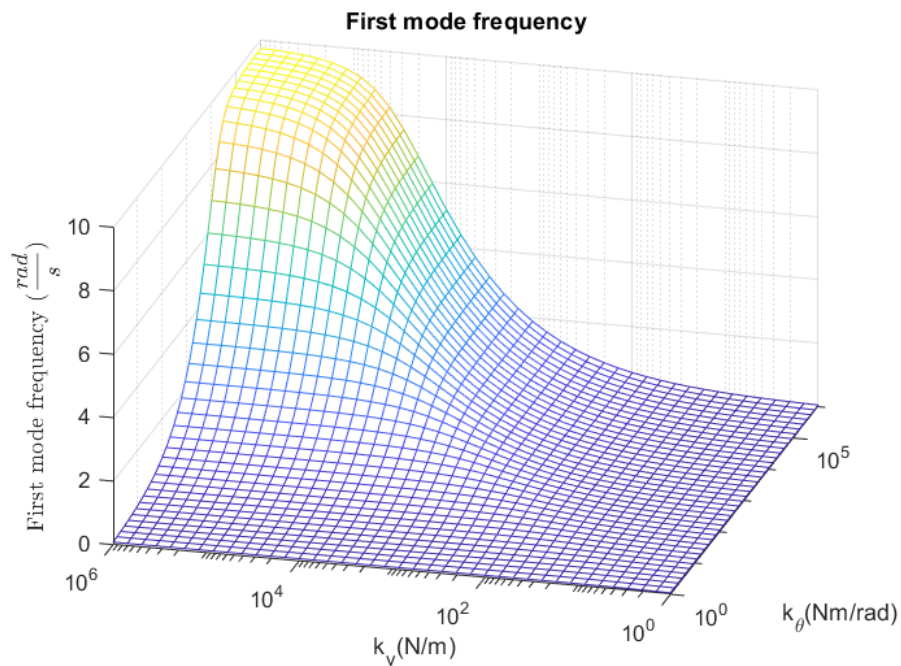
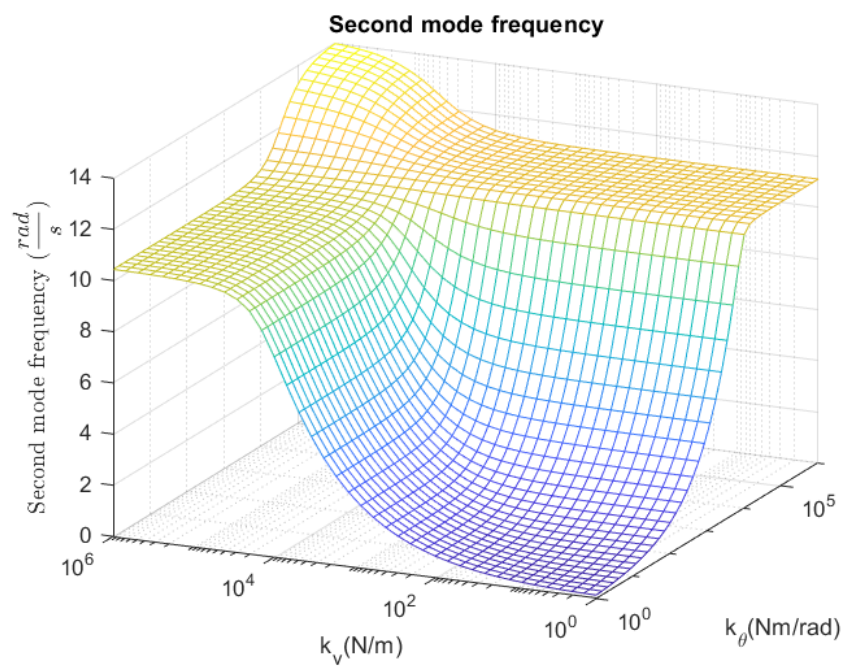
MPC	998	20001	1	1.	20005	1	-1.	Definition of the rigid motion of the Pod
MPC	998	20001	2	1.	20005	2	-1.	
MPC	998	20001	4	1.	20005	4	-1.	
MPC	998	20001	6	1.	20005	6	-1.	
CELAS2	2105	500.	20001	3	20005	3		Definition of the flexion spring
CELAS2	2106	500.	20001	5	20005	5		Definition of the torsional spring

Figure 5.2: Section of bdf file to model connection with springs

## 5.1 Effects of spring stiffness on natural frequencies.

In order to illustrate the effect of the springs on the structure, the first four natural frequencies have been calculated for a range of values of  $k_v$  and  $k_\theta$ . Figure 5.3 shows the frequencies for a given value of  $k_\theta$  and a wide range of  $k_v$ . Figure 5.4 and 5.5 show the effect of both stiffness on the first and second natural frequency.

Figure 5.3: First four modes Frequencies- $k_v$  for  $k_\theta = 10^6$

Figure 5.4: Influence of  $k_v$  and  $k_\theta$  on the first mode frequencyFigure 5.5: Influence of  $k_v$  and  $k_\theta$  on the second mode frequency

Frequencies have horizontal asymptotes for great values of  $k_v$  and  $k_\theta$ . Their asymptotic

values are the ones obtained in chapter 2, which means that by increasing the stiffness of the joints, the frequencies approach the values of the rigid joint case (first mode is plunge and second mode is torsion). For  $k_v \geq 10^5 N/m$  and  $k_\theta \geq 10^5 Nm/rad$ , the frequencies have values very close to the ones of the rigid joint configuration.

The lower values obtained correspond to the natural frequencies of the springs, that makes sense because for very low  $k$  values, the frequency mode is proportional to  $\sqrt{k}$ .

The intermediate values in the plot of second mode (flat part of the plot) correspond to the frequency of the plunge mode. This is because one stiffness is still very low and the first mode is approximately the natural frequency of that spring. However, as it was said, when both stiffness values are high enough, the frequencies of mode 2 now tend to that of the torsion mode, and the values of mode 1 tend to the plunge mode.

## 5.2 Study of pod vibrations.

One of the goals of this analysis report was to find the values for  $k_v$  and  $k_\theta$  so that the vibration amplitude of the pod would be smaller than the ones obtained at the wing. As it is shown below, this is not possible, as the vibrations of the pod are always bigger than those of the wing.

### 5.2.1 Effects of spring stiffness on the pod vibration amplitude.

The motion of the whole structure is influenced by the values of the stiffness. In the previous section, the influence on natural frequencies was studied. Now, the focus will be on the ratio of vibration amplitude between the tip of the wing and the pod. A comparison between displacements for grids 20001 (pod) and 20005 (tip of the wing) has been made for the first modes using different stiffness. These results are shown in figures 5.6 and 5.7 .

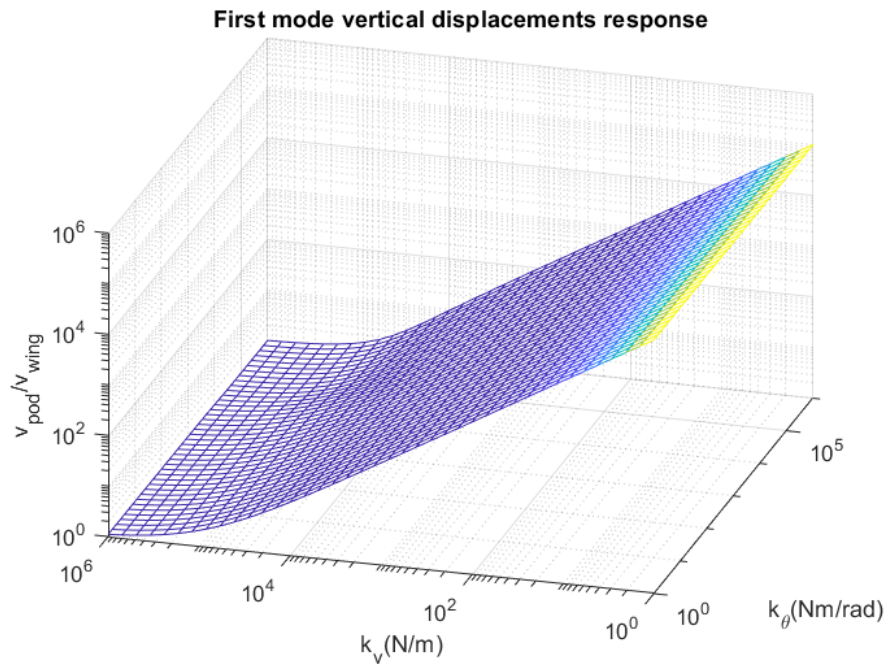


Figure 5.6: Ratio of vertical displacements in plunge mode as a function of  $k_v$  and  $k_\theta$

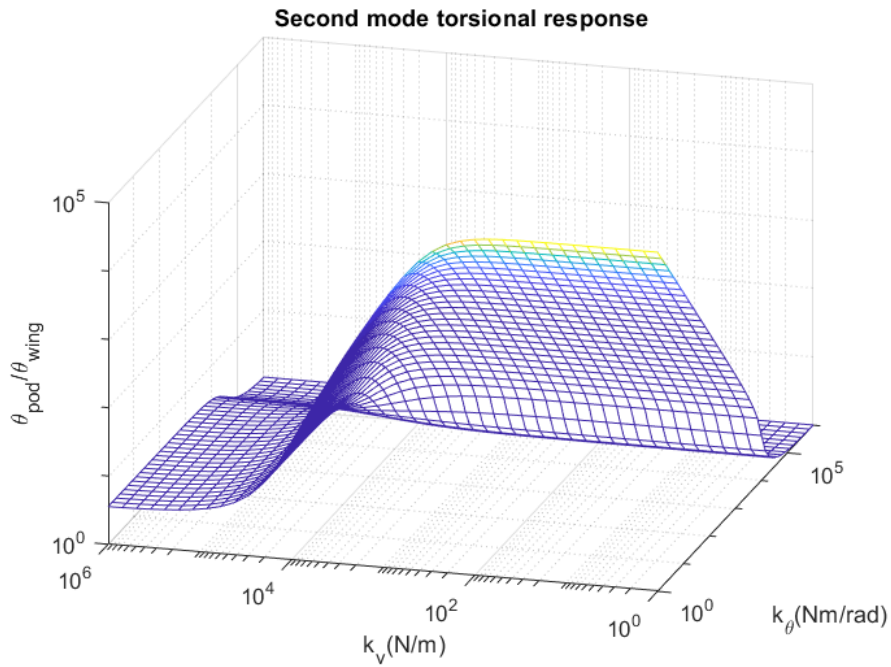


Figure 5.7: Ratio of spins in torsion mode as a function of  $k_v$  and  $k_\theta$

Both figures show that the pod motion will be greater in terms of vertical displacement and spin than the wing motion for any values of  $k_v$  and  $k_\theta$ . The minimum is found when both  $k_v$  and  $k_\theta$  are extremely high, which means rigid joint. In that case, both ratios are equal to 1.

Therefore, these springs have no use if the goal is to reduce the vibrations of the pod. However, they can be used to reduce wing vibrations and improve flutter behaviour as it will be explained in the next sections.

### 5.3 Analysis of wing vibrations.

There is plenty of information regarding the theory of dynamics in order to decrease vibrations on any system. One of the most popular approaches is the theory of *mass damper*, which could be used to reduce the vibration amplitude under certain conditions.

The wing-pod system can be regarded as one with a mass damper. In this case, the pod acts as the mass damper while the wing is the system whose vibrations need to be reduced.

#### 5.3.1 Mass damper.

Structures with small damping as this one, may develop vibrations of big amplitudes for loads acting at frequencies close to resonance. This response can be reduced by connecting a mass through a spring and a damper whose values shall be tuned. The original idea of the tuned mass damper belongs to Frahm [1], who did only include a spring but not a damper in his mass damper design.

In this section, a mass damper design will be proposed for the model. The pod will be attached with 2 springs but without dampers, since this is a design requirement. However, if a damper was introduced, it would be more effective reducing vibrations.

Figure 5.8, shows the configuration for a single degree of freedom problem.

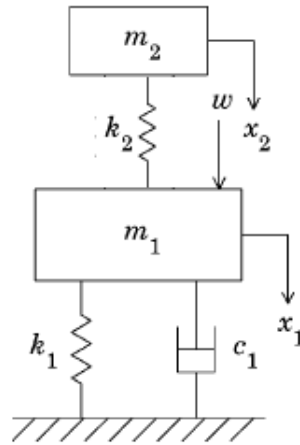


Figure 5.8: Dynamic vibration absorber attached to a single degree of freedom system [3]

In the studied case, the main body ( $m_1$ ) whose vibrations need to be reduced would be the wing, while the pod would act as the mass damper ( $m_2$ ).

A similar configuration was studied in [3], where the vibration amplitude is obtained as a function of the mass ratio ( $\mu = \frac{m_2}{m_1}$ ), the natural frequencies ( $\omega_1 = \sqrt{\frac{k_1}{m_1}}$  and  $\omega_2 = \sqrt{\frac{k_2}{m_2}}$ , the damping ratio ( $\xi_1 = \frac{c_1 \omega_1}{2k_1}$ ) and the external excitation frequency ( $\omega$ ).

A typical shape of this function has been obtained for the configuration shown in figure 5.8. The values of the parameters do not match with the ones of the wing and the pod studied. There is no need for that in order to show the qualitative influence of the mass damper on the system.

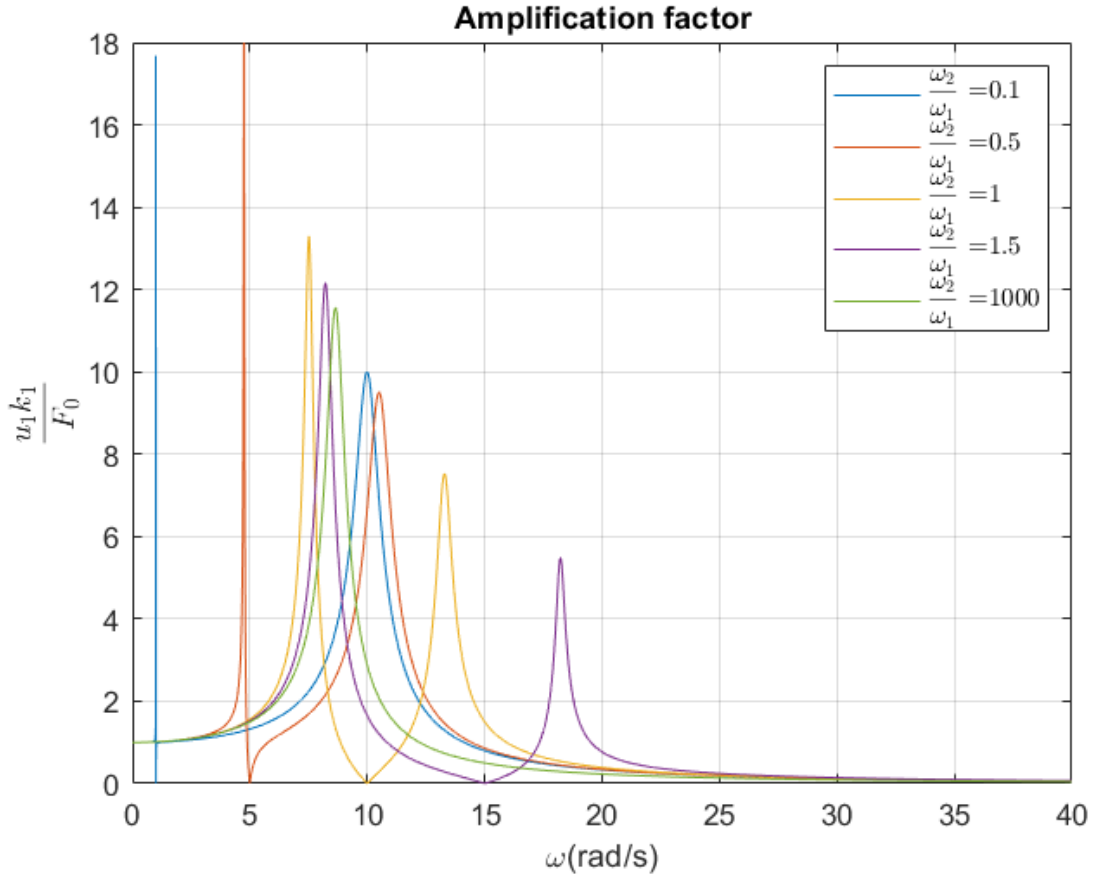


Figure 5.9: Amplification factor in a system with  $\mu = \frac{1}{3}$ ,  $\omega_1 = 10 \frac{rad}{s}$  and  $\xi_1 = 0.05$  for a range of  $\omega_2$  and  $\omega$

The figure shows that properly tuning the stiffness of the spring  $k_2$  (i.e. its natural frequency  $\omega_2$ ), the values of  $\omega$  where the function has a minimum or a maximum can be changed. The minimum amplification happens when the load acts near the natural frequency of the mass damper. This system proves to be very effective when the frequencies of the external loads are known.

### Using pod and springs as mass dampers.

If the pod was considered as a combination of two mass dampers, a torsional and a longitudinal one, their resonance frequencies could be easily tuned in order to match the unknown external load frequency.

The resonance frequency of a longitudinal spring mass damper is  $f = \frac{1}{2\pi} \sqrt{\frac{k_v}{m}}$ , while for a torsion spring it is  $f = \frac{1}{2\pi} \sqrt{\frac{k_\theta}{I}}$ . Where  $m$  is its mass and  $I$ , its moment of inertia.



The pod mass and inertia values are  $m_{pod} = 328.3kg$  and  $I_{pod} = 161.7kgm^2$ . Consequently, the desired values of stiffness to reduce vibrations would be  $k_v = (2\pi f)^2 m_{pod}$  and  $k_\theta = (2\pi f)^2 I_{pod}$ , with  $f$  the frequency of the external load.

This criteria of selecting stiffness values will be applied in section (6.1) to increase the flutter-boundary velocity.

## 5.4 Analysis of the first plunge and torsion modes. Wing and pod motion decoupling

Another criteria to tune the stiffness constants is presented in this subsection.

In order to understand how stiffness will affect the shape of the first modes of plunge and torsion, a representation of the free-wing (stiffness constants equal to zero) and joined-wing (rigidly attached to the pod) is shown in Figure 5.10 and Figure 5.11.

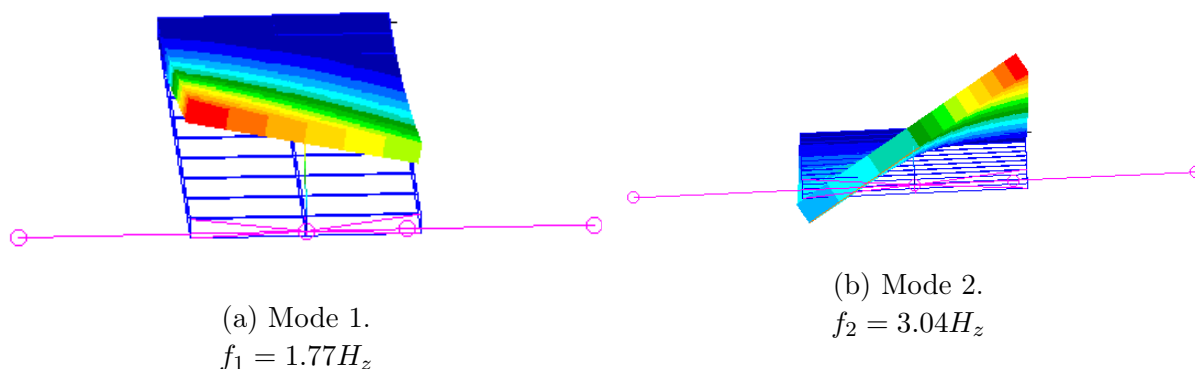


Figure 5.10: Free wing modes.

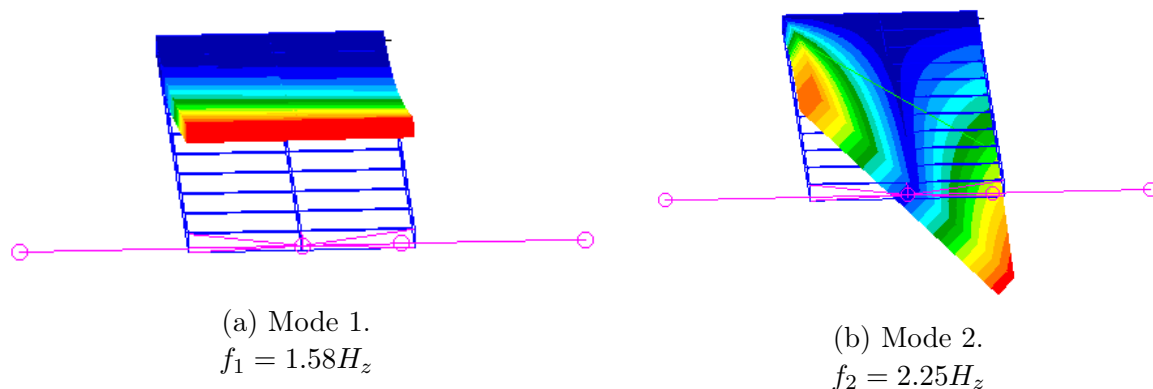


Figure 5.11: Rigid-joined wing modes.

Figure 5.10 shows that plunge and torsion are coupled for the first two modes of the free-wing. One design requirement could be to uncouple the first two modes of the wing so that there would be pure plunge for the first one and pure torsion for the second. To achieve this, their frequencies have to be close to the ones of the rigid joining case, which involves high values of stiffness.

Furthermore, an additional goal could be to uncouple the motion between wing and pod. This means that the vertical displacement and pitch of the pod would not be influenced by the plunge and torsion of the wing. This uncoupling condition can be achieved by maximizing the relative motion between pod and wing. And by recalling the results obtained in section 5.2.1, this relative motion was increased by setting low values of  $k_v$  and  $k_\theta$ .

In conclusion, in order to satisfy both of the mentioned requirements a compromise solution must be achieved. By trying to find intermediate stiffness values, and using the results from Figure 5.4 and Figure 5.5, a couple of stiffness constants that satisfy the frequency constraint are  $k_v = 3.75 \cdot 10^5 N/m$  and  $k_\theta = 3.75 \cdot 10^5 Nm/rad$ . It was proved that lower values of stiffness constants provided a first mode of pure plunge, but they degraded the response of the second mode, so they were discarded. The modal shapes using the selected stiffness constants are shown in Figure 5.12.

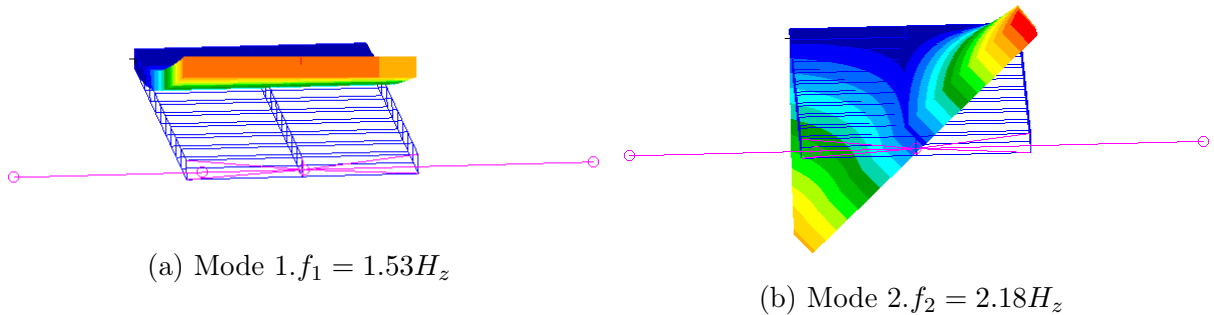


Figure 5.12: Natural modes of the wing-pod system

These results show that the rotation motion around the wing axis has been significantly damped in the first mode, resulting in a quasi-pure plunge mode. Additionally, the vertical displacement amplitude in the second mode has been reduced in contrast to the free-wing second mode. Regarding the natural frequencies, they are lower than the ones of the rigid joined wing (1.16% for first mode and 2.24% for the second one). Finally, the relative vibration amplitude between the wing's tip and the pod is  $\frac{v_{pod}}{v_{wing}} = 1.0929$  in the first mode and  $\frac{\theta_{pod}}{\theta_{wing}} = 1.0913$  in the second mode. This pair of stiffness constants provides a compromise solution for the mentioned requirements.

The influence of these results in  $V_{EAS}|_{flutter}$  will be discussed in section 6.1.

# Chapter 6

## Flutter Instability in the Flexible Joint Configuration

Spring stiffness values alter the response of the wing to flutter instability, as they modify the natural behaviour of the structure to free vibration as well as its interaction with the aerodynamic forces.

It is important to recall from the flutter analysis in the rigid joining case that flutter occurred at  $V_{EAS} = 65.8 \frac{m}{s}$  for the first normal mode, while at  $V_{EAS} = 93.9 \frac{m}{s}$  the dynamic divergence appeared for the second one.

Spring stiffness can be tuned in order to increase the flutter-boundary speed. In this section, the results of two different procedures are shown. The first one, is based on the concept of mass damper explained in section 5.3.1; the second one consists in uncoupling the torsion and flexion of the first two modes of the structure, explained in section 5.4. Each procedure returns two different values for  $k_v$  and  $k_\theta$ , and a different value of flutter boundary velocity.

### 6.1 Increasing flutter speed matching flutter and mass damper resonance frequency

In section 5.3.1, the use of the pod as a mass damper was explained and it was said that it was possible to reduce the vibrations of the system by tuning its resonance frequency. This was very useful when the frequency of the external load acting on the structure was known. Following this idea, in order to increase the velocity of the flutter boundary, the resonance frequency of the mass damper should be close to the frequency where flutter instability is expected to appear.

In the rigid attachment configuration, flutter instability was associated with a frequency

$f_{flutter} = 1.61Hz$ . The pod has a mass  $m_{pod} = 328.3kg$ . Therefore, in order to improve flutter, its stiffness should be  $k_v = m_{pod}(2\pi f_{flutter})^2 = 3.36 \cdot 10^4 N/m$ .

The torsional spring can be tuned in the same way. With  $I_{pod} = 161kgm^2$ ,  $k_\theta = I_{pod}(2\pi f_{flutter})^2 = 1.65 \cdot 10^4 Nm/rad$ .

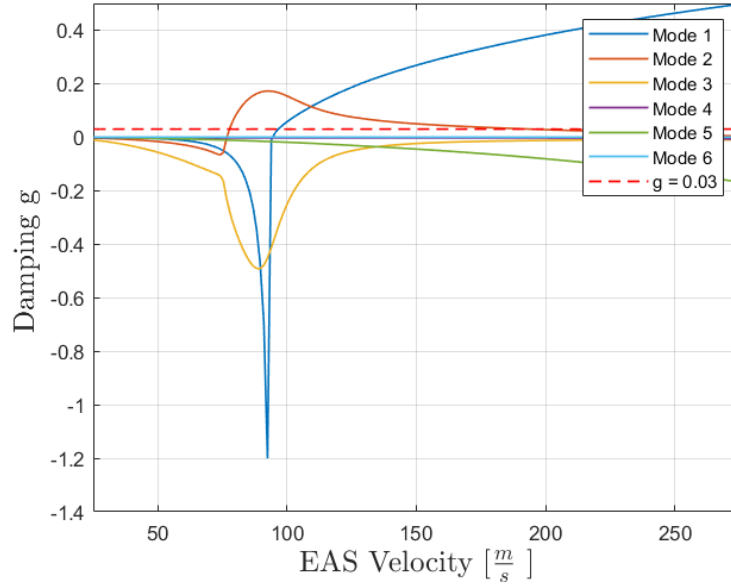


Figure 6.1: V-g Diagram with  $k_v = 3.36 \cdot 10^4 N/m$  and  $k_\theta = 1.65 \cdot 10^4 Nm/rad$

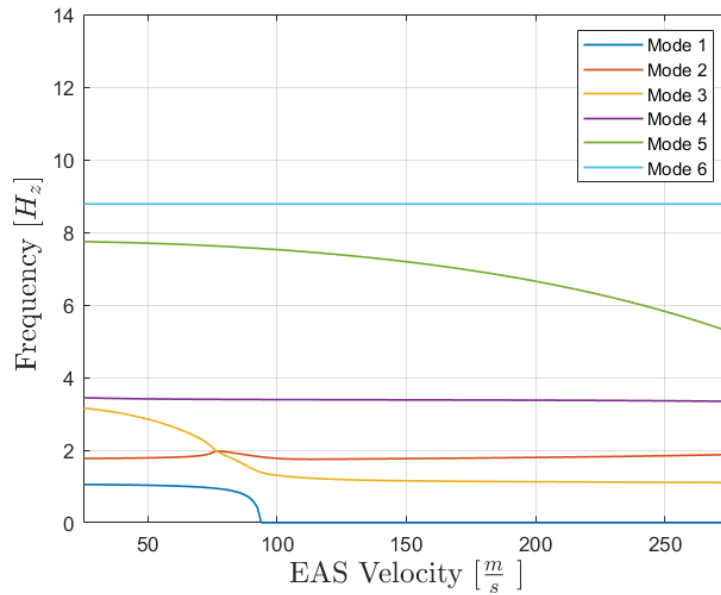


Figure 6.2: V-f Diagram with  $k_v = 3.3610^4 \cdot N/m$  and  $k_\theta = 1.65 \cdot 10^4 Nm/rad$

Figure 6.1 and 6.2 show that flutter is no longer associated with the first mode, but with the second one, at a velocity  $V_{EAS} = 77.7 \frac{m}{s}$ , that is greater than in the rigid case and at a frequency  $f_{flutter} = 2Hz$ .

By tuning again the mass damper for the new flutter frequency:  $k_v = 5.185 \cdot 10^4 N/m$  and  $k_\theta = 2.542 \cdot 10^4 Nm/rad$ .

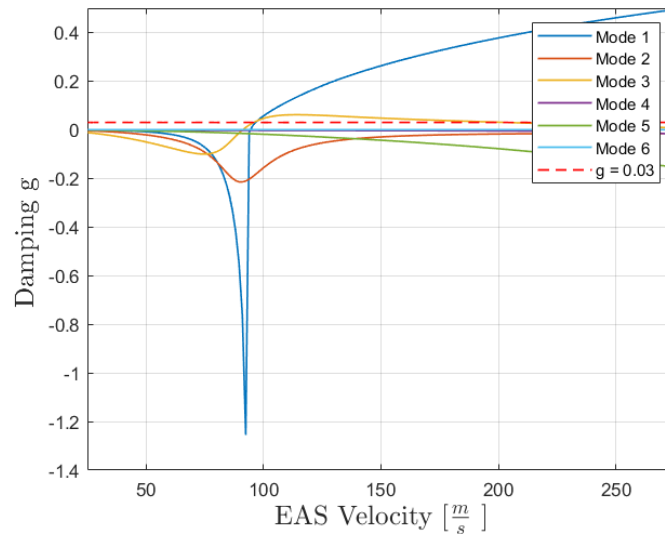


Figure 6.3: V-g Diagram with  $k_v = 5.185 \cdot 10^4 N/m$  and  $k_\theta = 2.542 \cdot 10^4 Nm/rad$

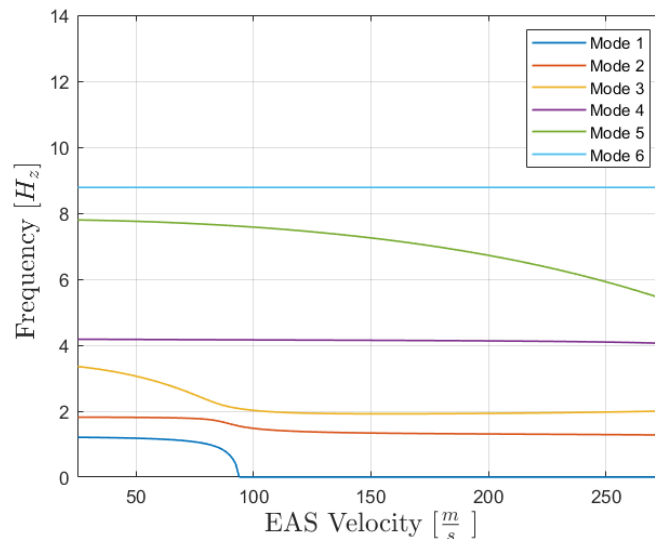


Figure 6.4: V-f Diagram with  $k_v = 5.185 \cdot 10^4 N/m$  and  $k_\theta = 2.542 \cdot 10^4 Nm/rad$

Flutter velocity has increased again. Now its value is  $95.7 \frac{m}{s}$ , almost the same as the divergence velocity, which is not affected by the spring stiffness. The flutter frequency for this case is  $2.1 Hz$ . Now flutter is associated with the third mode.

The third iteration is the last one, as it returns the same flutter frequency than the second one ( $f = 2.1 Hz$ ). The stiffness are  $k_v = 5.716 \cdot 10^4 N/m$  and  $k_\theta = 2.803 \cdot 10^4 Nm/rad$ . Its diagrams are shown below:

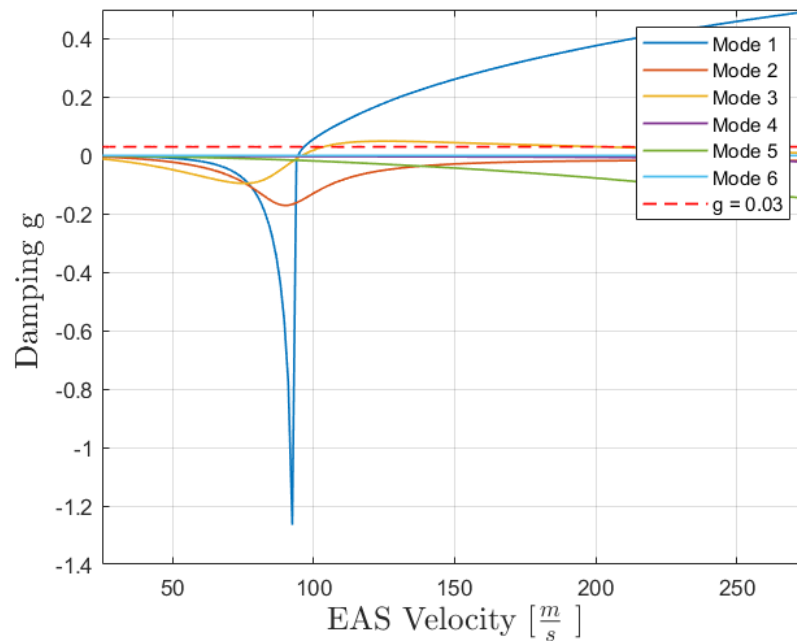


Figure 6.5: V-g Diagram with  $k_v = 5.716 \cdot 10^4 N/m$  and  $k_\theta = 2.803 \cdot 10^4 Nm/rad$

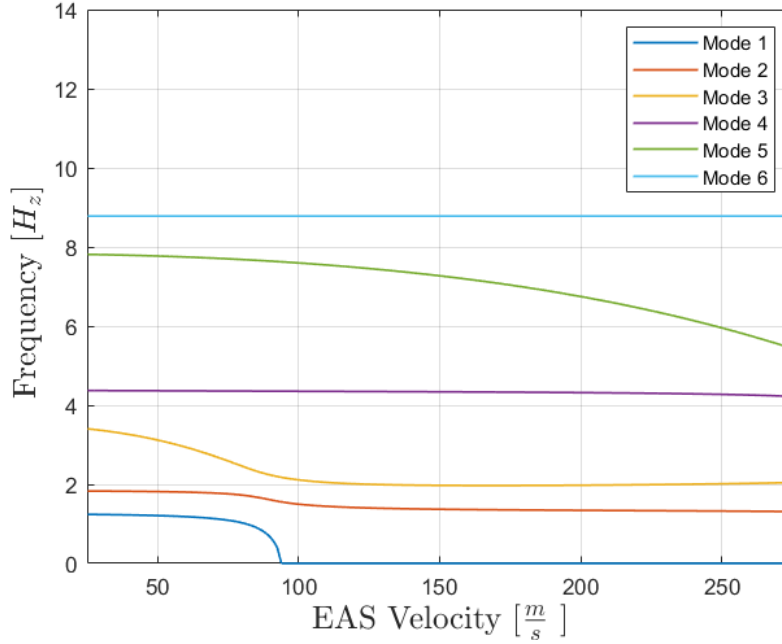


Figure 6.6: V-f Diagram with  $k_v = 5.716 \cdot 10^4 N/m$  and  $k_\theta = 2.803 \cdot 10^4 Nm/rad$

Flutter boundary appears at mode 3, at a velocity  $V_{EAS} = 103 \frac{m}{s}$ .

This procedure to tune springs has shown good results to increase flutter velocity. The values of  $k_v$  and  $k_\theta$  obtained are not the optimal ones for increasing it, as the relation between modes, frequencies and aerodynamic loading is complex and a more extended analysis would be necessary to find them. However they will be used as initial points for a sensitivity analysis of flutter-speed boundary.

As a general rule, the improvement the flutter is associated to the separation in frequency of the modes that exchange energy between them, as it can be seen in the previous graphs.

### 6.1.1 Frequency analysis

A frequency analysis has been carried out for the rigid joint configuration and for the wing with the values of stiffness obtained at the end of section 6.1 ( $k_v = 5.716 \cdot 10^4 N/m$  and  $k_\theta = 2.803 \cdot 10^4 Nm/rad$ ). The results obtained are the displacement of the external wing rib (grid 20005) when a load of frequency  $f$  is applied on that same node. SOL 111 of Nastran has been used for this purpose. Figure 6.7 shows how different the dynamic behaviour of both configurations is.

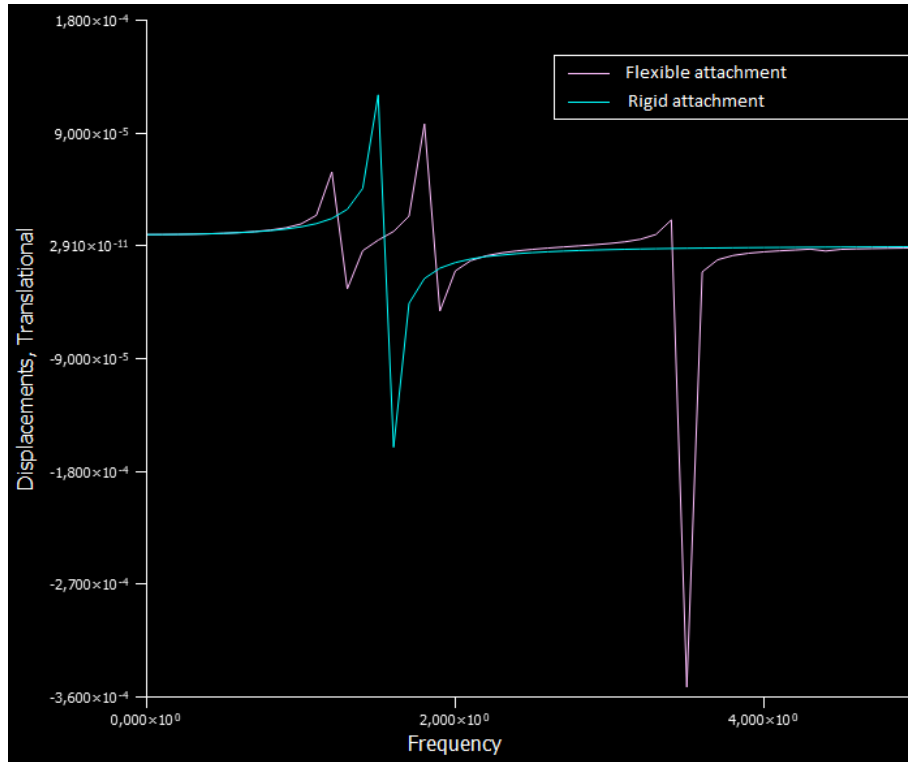


Figure 6.7: Relation between vertical displacement and frequency for the vertical load applied.

## 6.2 Influence of decoupling modes on flutter phenomena

In section 5.4, a pair of spring stiffness values were obtained following a criteria of decoupling torsional and plunge modes. In this section, it will be shown how the results of applying the PKNL method in Nastran with those stiffness ( $k_v = 3.75 \cdot 10^5 \frac{N}{m}$  and  $K_\theta = 3.75 \cdot 10^5 \frac{Nm}{rad}$ ) will have an influence on flutter phenomenon.

The Figure 6.8 showing the damping coefficient confirms a static divergence at the same point where the rigid-joined model also diverges, while flutter disappears completely.

Furthermore, it can be seen in Figures 6.8 and 6.9 how the second mode (red curve) tries to get into flutter by extracting energy from the first (blue curve) and third mode (yellow curve), which slowly decreases its damping and frequency. In the end, the frequencies of both modes 2 and 3 get to be separated from each other, which means that mode 2 does not get to extract the sufficient energy from mode 3 to get over the 3% damping line.

As a conclusion, it can be noticed that both springs not only manage to uncouple the normal modes but they also increase the flutter speed to an undefined limit.



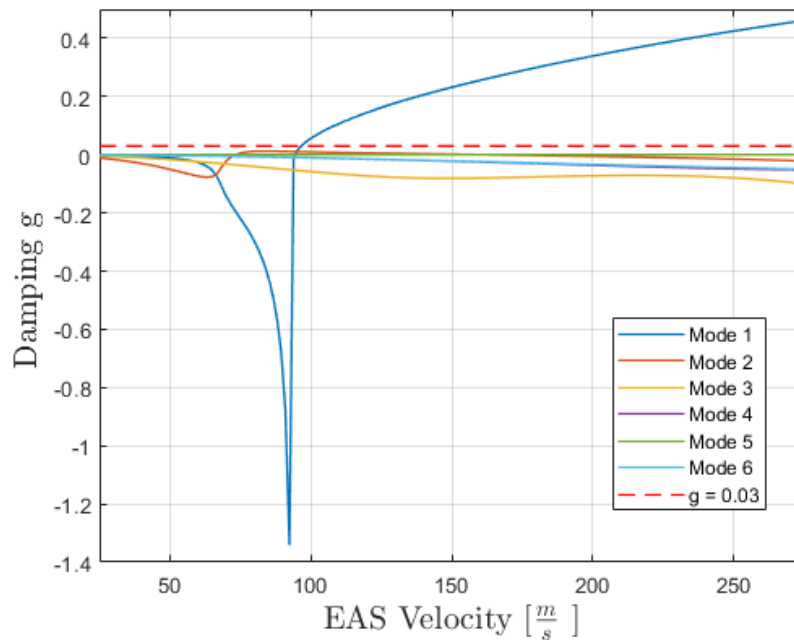


Figure 6.8: V-g diagram with the implemented springs

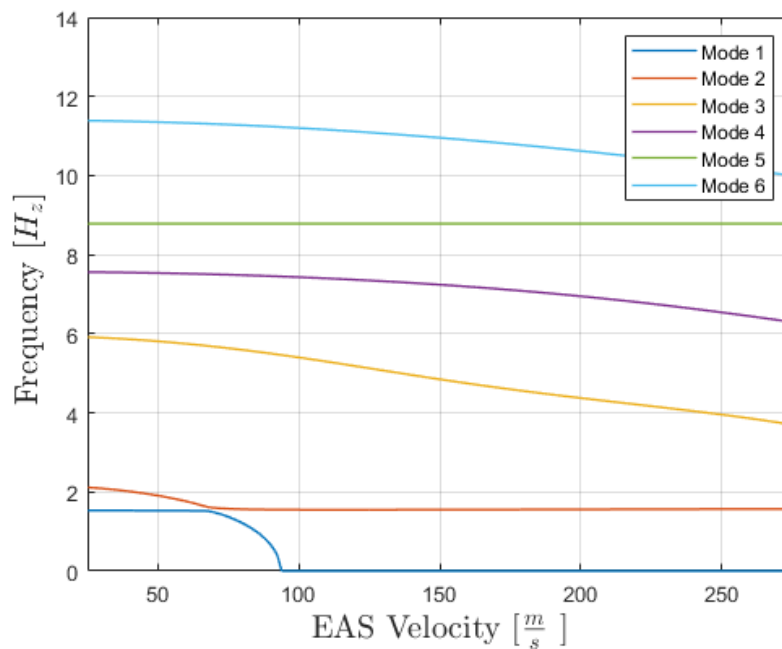


Figure 6.9: V-f diagram with the implemented springs

### 6.3 Chosen spring stiffness values

In the previous sections 6.1 and 6.2, stability boundaries for two different pairs of spring stiffness were studied. The pair obtained in section 6.1 increases flutter velocity by 56%, which is a significant improvement in terms of stability. The other pair, obtained in 5.4, has shown an even better improvement, as flutter does not occur for the studied flight conditions. Therefore, these last ones will be chosen as the final values for the flexible joint that will satisfy the mentioned design requirements ( $k_v = 3.75 \cdot 10^5 N/m$  and  $k_\theta = 3.75 \cdot 10^5 Nm/rad$ ).

# Chapter 7

## Sensitivity Analysis

It has been proved from section 6.2 that there were values of  $k_v$  and  $k_\theta$  for which not only could the first normal modes of the wing be uncoupled but they also managed to avoid flutter at  $M = 0.8$ .

However, if these stiffness values were to be used as the central points within a range of  $k_v$  and  $k_\theta$  to find how flutter velocities vary in a sensitivity analysis, the latter could not be performed since flutter does not appear as mentioned above.

Indeed, for this sensitivity analysis, the pair of  $k_v$  and  $k_\theta$  that have been used are the ones obtained in section 6.1, in which the concept of *mass damper* was used in order to find stiffness values that would progressively increase the flutter-boundary velocity. These values are:  $k_v = 5.716 \cdot 10^4 \frac{N}{m}$  and  $k_\theta = 2.803 \frac{Nm}{rad}$

The analysis was set up to express flutter speed as a function of the torsion spring constant,  $k_\theta$ , in which  $k_v$  remained as variable parameter. By running several simulations the results obtained are shown in figure 7.1.

Notice how flutter speed always grows with both spring constants in the range selected.

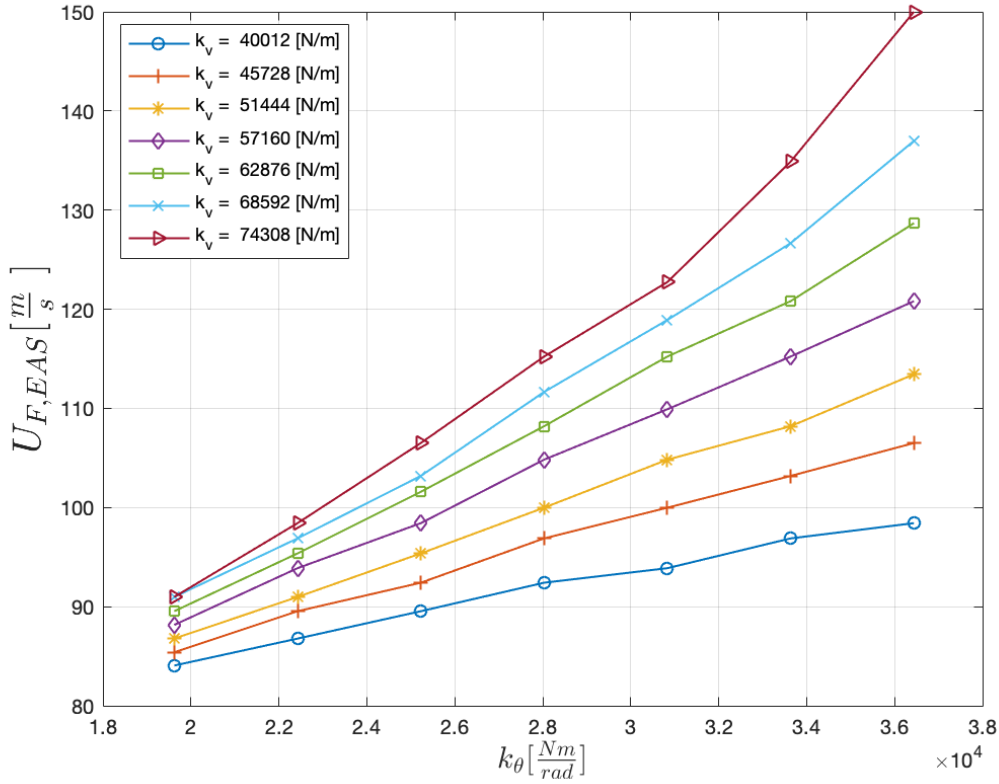


Figure 7.1: Evolution of flutter-boundary speed as a function of  $k_v$ , for different constant values of  $k_\theta$

Increasing stiffness along the gradient direction might result in obtaining an optimal pair of values. There is no need to do that, as a pair of stiffness values that avoid flutter has already been obtained. Nevertheless, this figure shows valuable information, as for example, how sensitive flutter velocity  $U_{F,EAS}$  is to changes in stiffness.

However, the behaviour of the system is not monotonous as it can be expected with figure 7.1. When high enough values are reached, the flutter velocity decreases again. This fact is proved because when both  $k$  tend to infinite (rigid case), the flutter velocity tends to  $65.8m/s$ .

# Chapter 8

## Concluding Remarks

The present work has accomplished all its objectives. Firstly, the structure has been characterized by its first 16 normal modes (the ones with a frequency below  $50Hz$ ). In addition, its generalized mass and stiffness matrices have been calculated.

After describing the flutter equation for the K-Method, a flutter analysis with Nastran has been carried out, using the PK method. As a result, the instability boundaries of the wing (flutter and divergence) have been obtained. There was a big difference between the flutter-boundary velocity and divergence velocity. The range of operation is limited by the lowest of these two (flutter in this case).

The suggested structural alteration of the wing, which consisted in changing the rigid joint between wing and pod into a flexible one, was successfully implemented in the model. The stiffness values of the springs used in the flexible union were used as design variables in order to reduce wing vibrations.

An initial requirement of design was to reduce pod vibrations related to those of the wing. However, this report proved that it was not possible. The configuration under which they are minimum is the rigid case.

Nevertheless, wing vibrations could be reduced by properly tuning the spring stiffness based on the *mass damper* concept. By doing so, flutter-instability velocity could be increased.

A second requirement was to uncouple motion between wing and pod, which has also been achieved.

All together, two different approaches were made in order to correctly set the spring stiffness. Both have shown very good results; the first one increased flutter velocity by 56%, while the other makes flutter to disappear.

Hence, the modified wing range of operation is no longer limited by flutter, as the most restrictive speed is the one for which divergence appears, on which flexible joining has

no effect. If the design goal was to increase this range, the divergence speed should be increased.

A way to do this without altering the structure, is by choosing a material with an higher Young's module. The ultimate configuration will be able to fly at higher speeds and lower altitudes than the initial one. These changes could make the wing more valuable to potential customers as it could be used in a more extended flight envelope, turning it into a more versatile wing.



# Chapter 9

## Appendix

### 9.1 Bdf file for flutter analysis of rigid joint configuration

```

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2  $                               SOL 145 - flutter Analysis
3  $ =====
4  $
5  $ ----- File Management Section -----
6  $ ASSIGN OUTPUT4 = 'btb.f50',UNIT = 50,FORM=FORMATTED
7  $
8  $ ----- Executive Control Deck -----
9  $
10 ID                MSC.NASTRAN FLUTTER ANALYSIS
11 SOL                145 ← Flutter Solution
12 TIME              5000
13 $
14 $ ----- Executive Control Deck -----
15 $
16 CEND
17 $
18 $ ----- Case Control Deck -----
19 $
20 TITLE              = FLUTTER ANALYSIS
21 SUBTITLE           = (SOL145)
22 $
23 ECHO                = NONE
24 SEALL               = ALL
25 SPC                 = 999
26 METHOD              = 200 ← Real Eigenvalue Method Selection
27 FMETHOD           = 300 ← Flutter Analysis Method Selection
28 RESVEC              = NO
29 DISPLACEMENT      = ALL
30 $
31 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
32 PARAM POST         0
33 PARAM GRDPNT       0
34 PARAM OPHIPA      1
35 PARAM PRTMAXIM    YES
36 PARAM OPPHIB      1
37
38 BEGIN BULK
39 $ MATERIAL QUE USA PARA LAS BARRAS (PROD) Y PARA LAS CHAPAS O REVESTIMIENTO (PSHELL)
40 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
41 MAT1 1 7.171+102.689+10 5.1538-4
42 $
43 $ GRIDS DEL LA LINEA MORADA DE LA RAIZ DEL ALA:POD
44 GRID 20001 0.6096 6.2484 0. 123456
45 GRID 20002 0.0762 6.2484 0.
46 GRID 20003 -0.9144 6.2484 0.
47 GRID 20004 2.1336 6.2484 0.
48 $ CREO LAS LINEAS MORADAS: BARRAS RIGIDAS
49 RBAR 3001 20001 20002 123456
50 RBAR 3002 20002 20003 123456
51 RBAR 3003 20001 20004 123456
52 $
53
54
55 $ GRIDS QUE FORMAN EL ALA, A PARTIR DE LOS CUALES SE DEFINEN LOS CROD Y CQUAD4

```



```

73  GRID      10204      0.6096  1.2192  -.0508
74  GRID      10205      1.2192  1.2192  -.0508
75  GRID      10300      0.        1.8288  .0508
76  GRID      10301      0.6096  1.8288  .0508
77  GRID      10302      1.2192  1.8288  .0508
78  GRID      10303      0.        1.8288  -.0508
79  GRID      10304      0.6096  1.8288  -.0508
80  GRID      10305      1.2192  1.8288  -.0508
81  GRID      10400      0.        2.4384  .0508
82  GRID      10401      0.6096  2.4384  .0508
83  GRID      10402      1.2192  2.4384  .0508
84  GRID      10403      0.        2.4384  -.0508
85  GRID      10404      0.6096  2.4384  -.0508
86  GRID      10405      1.2192  2.4384  -.0508
87  GRID      10500      0.        3.048   .0508
88  GRID      10501      0.6096  3.048   .0508
89  GRID      10502      1.2192  3.048   .0508
90  GRID      10503      0.        3.048   -.0508
91  GRID      10504      0.6096  3.048   -.0508
92  GRID      10505      1.2192  3.048   -.0508
93  GRID      10600      0.        3.6576  .0508
94  GRID      10601      0.6096  3.6576  .0508
95  GRID      10602      1.2192  3.6576  .0508
96  GRID      10603      0.        3.6576  -.0508
97  GRID      10604      0.6096  3.6576  -.0508
98  GRID      10605      1.2192  3.6576  -.0508
99  GRID      10700      0.        4.2672  .0508
100 GRID      10701      0.6096  4.2672  .0508
101 GRID      10702      1.2192  4.2672  .0508
102 GRID      10703      0.        4.2672  -.0508
103 GRID      10704      0.6096  4.2672  -.0508
104 GRID      10705      1.2192  4.2672  -.0508
105 GRID      10800      0.        4.8768  .0508
106 GRID      10801      0.6096  4.8768  .0508
107 GRID      10802      1.2192  4.8768  .0508
108 GRID      10803      0.        4.8768  -.0508
109 GRID      10804      0.6096  4.8768  -.0508
110 GRID      10805      1.2192  4.8768  -.0508
111 GRID      10900      0.        5.4864  .0508
112 GRID      10901      0.6096  5.4864  .0508
113 GRID      10902      1.2192  5.4864  .0508
114 GRID      10903      0.        5.4864  -.0508
115 GRID      10904      0.6096  5.4864  -.0508
116 GRID      10905      1.2192  5.4864  -.0508
117 GRID      11000      0.        6.096   .0508
118 GRID      11001      0.6096  6.096   .0508
119 GRID      11002      1.2192  6.096   .0508
120 GRID      11003      0.        6.096   -.0508
121 GRID      11004      0.6096  6.096   -.0508
122 GRID      11005      1.2192  6.096   -.0508
123 $
124 GRID      12000      1.8288  0.        .0508
125 SPC1     999      123456  12000
126
127
128 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
129 SPC      999      10000      1
130 SPC      999      10000      2
131 SPC      999      10000      3
132 SPC      999      10000      4
133 SPC      999      10000      5
134 SPC      999      10000      6
135 SPC      999      10001      1
136 SPC      999      10001      2
137 SPC      999      10001      3
138 SPC      999      10001      4
139 SPC      999      10001      5
140 SPC      999      10001      6
141 SPC      999      10002      1
142 SPC      999      10002      2
143 SPC      999      10002      3
144 SPC      999      10002      4

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145	SPC	999	10002	5
146	SPC	999	10002	6
147	SPC	999	10003	1
148	SPC	999	10003	2
149	SPC	999	10003	3
150	SPC	999	10003	4
151	SPC	999	10003	5
152	SPC	999	10003	6
153	SPC	999	10004	1
154	SPC	999	10004	2
155	SPC	999	10004	3
156	SPC	999	10004	4
157	SPC	999	10004	5
158	SPC	999	10004	6
159	SPC	999	10005	1
160	SPC	999	10005	2
161	SPC	999	10005	3
162	SPC	999	10005	4
163	SPC	999	10005	5
164	SPC	999	10005	6
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167	SPC	999	10100	6
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169	SPC	999	10101	5
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171	SPC	999	10102	4
172	SPC	999	10102	5
173	SPC	999	10102	6
174	SPC	999	10103	4
175	SPC	999	10103	5
176	SPC	999	10103	6
177	SPC	999	10104	4
178	SPC	999	10104	5
179	SPC	999	10104	6
180	SPC	999	10105	4
181	SPC	999	10105	5
182	SPC	999	10105	6
183	SPC	999	10200	4
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185	SPC	999	10200	6
186	SPC	999	10201	4
187	SPC	999	10201	5
188	SPC	999	10201	6
189	SPC	999	10202	4
190	SPC	999	10202	5
191	SPC	999	10202	6
192	SPC	999	10203	4
193	SPC	999	10203	5
194	SPC	999	10203	6
195	SPC	999	10204	4
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197	SPC	999	10204	6
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201	SPC	999	10300	4
202	SPC	999	10300	5
203	SPC	999	10300	6
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212	SPC	999	10303	6
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214	SPC	999	10304	5
215	SPC	999	10304	6
216	SPC	999	10305	4

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217	SPC	999	10305	5
218	SPC	999	10305	6
219	SPC	999	10400	4
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234	SPC	999	10405	4
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239	SPC	999	10500	6
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241	SPC	999	10501	5
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270	SPC	999	10605	4
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272	SPC	999	10605	6
273	SPC	999	10700	4
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300	SPC	999	10803	4
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302	SPC	999	10803	6
303	SPC	999	10804	4
304	SPC	999	10804	5
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309	SPC	999	10900	4
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311	SPC	999	10900	6
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321	SPC	999	10904	4
322	SPC	999	10904	5
323	SPC	999	10904	6
324	SPC	999	10905	4
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326	SPC	999	10905	6
327	SPC	999	11000	4
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330	SPC	999	11001	4
331	SPC	999	11001	5
332	SPC	999	11001	6
333	SPC	999	11002	4
334	SPC	999	11002	5
335	SPC	999	11002	6
336	SPC	999	11003	4
337	SPC	999	11003	5
338	SPC	999	11003	6
339	SPC	999	11004	4
340	SPC	999	11004	5
341	SPC	999	11004	6
342	SPC	999	11005	4
343	SPC	999	11005	5
344	SPC	999	11005	6
345				
346				
347	\$...1... ...2... ...3... ...4... ...5... ...6... ...7... ...8... ...9... ...10...			
348	RBE3	2100	20001	123456 1.0 123 11000 11001 +6001
349	+6001	11002	11003	11004 11005
350				
351	\$...1... ...2... ...3... ...4... ...5... ...6... ...7... ...8... ...9... ...10...			
352	PROD	5	1	2.5000-5
353	\$			
354	CROD	100	5	10003 10000
355	CROD	101	5	10004 10001
356	CROD	102	5	10005 10002
357	CROD	103	5	10103 10100
358	CROD	104	5	10104 10101
359	CROD	105	5	10105 10102
360	CROD	106	5	10203 10200

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361 CROD 107 5 10204 10201
362 CROD 108 5 10205 10202
363 CROD 109 5 10303 10300
364 CROD 110 5 10304 10301
365 CROD 111 5 10305 10302
366 CROD 112 5 10403 10400
367 CROD 113 5 10404 10401
368 CROD 114 5 10405 10402
369 CROD 115 5 10503 10500
370 CROD 116 5 10504 10501
371 CROD 117 5 10505 10502
372 CROD 118 5 10603 10600
373 CROD 119 5 10604 10601
374 CROD 120 5 10605 10602
375 CROD 121 5 10703 10700
376 CROD 122 5 10704 10701
377 CROD 123 5 10705 10702
378 CROD 124 5 10803 10800
379 CROD 125 5 10804 10801
380 CROD 126 5 10805 10802
381 CROD 127 5 10903 10900
382 CROD 128 5 10904 10901
383 CROD 129 5 10905 10902
384 CROD 130 5 11003 11000
385 CROD 131 5 11004 11001
386 CROD 132 5 11005 11002
387 $
388 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
389 PROD 6 1 2.5000-3
390 $
391 CROD 133 6 10000 10001
392 CROD 134 6 10001 10002
393 CROD 135 6 10003 10004
394 CROD 136 6 10004 10005
395 CROD 137 6 10100 10101
396 CROD 138 6 10101 10102
397 CROD 139 6 10103 10104
398 CROD 140 6 10104 10105
399 CROD 141 6 10200 10201
400 CROD 142 6 10201 10202
401 CROD 143 6 10203 10204
402 CROD 144 6 10204 10205
403 CROD 145 6 10300 10301
404 CROD 146 6 10301 10302
405 CROD 147 6 10303 10304
406 CROD 148 6 10304 10305
407 CROD 149 6 10400 10401
408 CROD 150 6 10401 10402
409 CROD 151 6 10403 10404
410 CROD 152 6 10404 10405
411 CROD 153 6 10500 10501
412 CROD 154 6 10500 10502
413 CROD 155 6 10503 10504
414 CROD 156 6 10504 10505
415 CROD 157 6 10600 10601
416 CROD 158 6 10601 10602
417 CROD 159 6 10603 10604
418 CROD 160 6 10604 10605
419 CROD 161 6 10700 10701
420 CROD 162 6 10701 10702
421 CROD 163 6 10703 10704
422 CROD 164 6 10704 10705
423 CROD 165 6 10800 10801
424 CROD 166 6 10801 10802
425 CROD 167 6 10803 10804
426 CROD 168 6 10804 10805
427 CROD 169 6 10900 10901
428 CROD 170 6 10901 10902
429 CROD 171 6 10903 10904
430 CROD 172 6 10905 10904
431 CROD 173 6 11000 11001
432 CROD 174 6 11001 11002

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433	CROD	175	6	11003	11004						
434	CROD	176	6	11004	11005						
435	\$										
436	\$	1	2	3	4	5	6	7	8	9	10
437	PROD	7	1	2.5000	-3						
438	\$										
439	CROD	177	7	10000	10100						
440	CROD	178	7	10003	10103						
441	CROD	179	7	10100	10200						
442	CROD	180	7	10103	10203						
443	CROD	181	7	10200	10300						
444	CROD	182	7	10203	10303						
445	CROD	183	7	10300	10400						
446	CROD	184	7	10303	10403						
447	CROD	185	7	10400	10500						
448	CROD	186	7	10403	10503						
449	CROD	187	7	10500	10600						
450	CROD	188	7	10503	10603						
451	CROD	189	7	10600	10700						
452	CROD	190	7	10603	10703						
453	CROD	191	7	10700	10800						
454	CROD	192	7	10703	10803						
455	CROD	193	7	10800	10900						
456	CROD	194	7	10803	10903						
457	CROD	195	7	10900	11000						
458	CROD	196	7	10903	11003						
459	CROD	197	7	10002	10102						
460	CROD	198	7	10005	10105						
461	CROD	199	7	10102	10202						
462	CROD	200	7	10105	10205						
463	CROD	201	7	10202	10302						
464	CROD	202	7	10205	10305						
465	CROD	203	7	10302	10402						
466	CROD	204	7	10305	10405						
467	CROD	205	7	10402	10502						
468	CROD	206	7	10405	10505						
469	CROD	207	7	10502	10602						
470	CROD	208	7	10505	10605						
471	CROD	209	7	10602	10702						
472	CROD	210	7	10605	10705						
473	CROD	211	7	10702	10802						
474	CROD	212	7	10705	10805						
475	CROD	213	7	10802	10902						
476	CROD	214	7	10805	10905						
477	CROD	215	7	10902	11002						
478	CROD	216	7	10905	11005						
479	\$										
480	\$	1	2	3	4	5	6	7	8	9	10
481	PROD	8	1	1.5000	-2						
482	\$										
483	CROD	217	8	10001	10101						
484	CROD	218	8	10004	10104						
485	CROD	219	8	10101	10201						
486	CROD	220	8	10104	10204						
487	CROD	221	8	10201	10301						
488	CROD	222	8	10204	10304						
489	CROD	223	8	10301	10401						
490	CROD	224	8	10304	10404						
491	CROD	225	8	10401	10501						
492	CROD	226	8	10404	10504						
493	CROD	227	8	10501	10601						
494	CROD	228	8	10504	10604						
495	CROD	229	8	10601	10701						
496	CROD	230	8	10604	10704						
497	CROD	231	8	10701	10801						
498	CROD	232	8	10704	10804						
499	CROD	233	8	10801	10901						
500	CROD	234	8	10804	10904						
501	CROD	235	8	10901	11001						
502	CROD	236	8	10904	11004						
503	\$										
504	\$	1	2	3	4	5	6	7	8	9	10

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505 $ DEFINE EL ESPESOR DE LA CHAPA 0.0025
506 PSHELL 1 1 .0025000
507 $
508 CQUAD4 1 1 10000 10001 10101 10100
509 CQUAD4 2 1 10001 10002 10102 10101
510 CQUAD4 3 1 10100 10101 10201 10200
511 CQUAD4 4 1 10101 10102 10202 10201
512 CQUAD4 5 1 10200 10201 10301 10300
513 CQUAD4 6 1 10201 10202 10302 10301
514 CQUAD4 7 1 10300 10301 10401 10400
515 CQUAD4 8 1 10301 10302 10402 10401
516 CQUAD4 9 1 10400 10401 10501 10500
517 CQUAD4 10 1 10401 10402 10502 10501
518 CQUAD4 11 1 10500 10501 10601 10600
519 CQUAD4 12 1 10501 10502 10602 10601
520 CQUAD4 13 1 10600 10601 10701 10700
521 CQUAD4 14 1 10601 10602 10702 10701
522 CQUAD4 15 1 10700 10701 10801 10800
523 CQUAD4 16 1 10701 10702 10802 10801
524 CQUAD4 17 1 10800 10801 10901 10900
525 CQUAD4 18 1 10801 10802 10902 10901
526 CQUAD4 19 1 10900 10901 11001 11000
527 CQUAD4 20 1 10901 10902 11002 11001
528 CQUAD4 21 1 10003 10004 10104 10103
529 CQUAD4 22 1 10004 10005 10105 10104
530 CQUAD4 23 1 10103 10104 10204 10203
531 CQUAD4 24 1 10104 10105 10205 10204
532 CQUAD4 25 1 10203 10204 10304 10303
533 CQUAD4 26 1 10204 10205 10305 10304
534 CQUAD4 27 1 10303 10304 10404 10403
535 CQUAD4 28 1 10304 10305 10405 10404
536 CQUAD4 29 1 10403 10404 10504 10503
537 CQUAD4 30 1 10404 10405 10505 10504
538 CQUAD4 31 1 10503 10504 10604 10603
539 CQUAD4 32 1 10504 10505 10605 10604
540 CQUAD4 33 1 10603 10604 10704 10703
541 CQUAD4 34 1 10604 10605 10705 10704
542 CQUAD4 35 1 10703 10704 10804 10803
543 CQUAD4 36 1 10704 10705 10805 10804
544 CQUAD4 37 1 10803 10804 10904 10903
545 CQUAD4 38 1 10804 10805 10905 10904
546 CQUAD4 39 1 10903 10904 11004 11003
547 CQUAD4 40 1 10904 10905 11005 11004
548 $
549 $. .1. . | . .2. . | . .3. . | . .4. . | . .5. . | . .6. . | . .7. . | . .8. . | . .9. . | . .10. .
550 PSHELL 2 1 1.0000-4
551 $
552 CQUAD4 41 2 10003 10000 10100 10103
553 CQUAD4 42 2 10103 10100 10200 10203
554 CQUAD4 43 2 10203 10200 10300 10303
555 CQUAD4 44 2 10300 10400 10403 10303
556 CQUAD4 45 2 10400 10500 10503 10403
557 CQUAD4 46 2 10500 10600 10603 10503
558 CQUAD4 47 2 10600 10700 10703 10603
559 CQUAD4 48 2 10700 10800 10803 10703
560 CQUAD4 49 2 10800 10900 10903 10803
561 CQUAD4 50 2 10900 11000 11003 10903
562 CQUAD4 51 2 10002 10102 10105 10005
563 CQUAD4 52 2 10102 10202 10205 10105
564 CQUAD4 53 2 10202 10302 10305 10205
565 CQUAD4 54 2 10302 10402 10405 10305
566 CQUAD4 55 2 10402 10502 10505 10405
567 CQUAD4 56 2 10502 10602 10605 10505
568 CQUAD4 57 2 10602 10702 10705 10605
569 CQUAD4 58 2 10702 10802 10805 10705
570 CQUAD4 59 2 10802 10902 10905 10805
571 CQUAD4 60 2 10902 11002 11005 10905
572 $
573 $. .1. . | . .2. . | . .3. . | . .4. . | . .5. . | . .6. . | . .7. . | . .8. . | . .9. . | . .10. .
574 PSHELL 3 1 .017500
575 $
576 CQUAD4 61 3 10004 10001 10101 10104

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577	CQUAD4	62	3	10104	10101	10201	10204
578	CQUAD4	63	3	10204	10201	10301	10304
579	CQUAD4	64	3	10304	10301	10401	10404
580	CQUAD4	65	3	10404	10401	10501	10504
581	CQUAD4	66	3	10504	10501	10601	10604
582	CQUAD4	67	3	10604	10601	10701	10704
583	CQUAD4	68	3	10704	10701	10801	10804
584	CQUAD4	69	3	10804	10801	10901	10904
585	CQUAD4	70	3	10904	10901	11001	11004
586	\$						
587	\$...1... ...2... ...3... ...4... ...5... ...6... ...7... ...8... ...9... ...10...						
588	PSHELL	4	1	.017500			
589	\$						
590	CQUAD4	71	4	10003	10000	10001	10004
591	CQUAD4	72	4	10004	10001	10002	10005
592	CQUAD4	73	4	10103	10100	10101	10104
593	CQUAD4	74	4	10104	10101	10102	10105
594	CQUAD4	75	4	10203	10200	10201	10204
595	CQUAD4	76	4	10204	10201	10202	10205
596	CQUAD4	77	4	10303	10300	10301	10304
597	CQUAD4	78	4	10304	10301	10302	10305
598	CQUAD4	79	4	10403	10400	10401	10404
599	CQUAD4	80	4	10404	10401	10402	10405
600	CQUAD4	81	4	10503	10500	10501	10504
601	CQUAD4	82	4	10504	10501	10502	10505
602	CQUAD4	83	4	10603	10600	10601	10604
603	CQUAD4	84	4	10604	10601	10602	10605
604	CQUAD4	85	4	10703	10700	10701	10704
605	CQUAD4	86	4	10704	10701	10702	10705
606	CQUAD4	87	4	10803	10800	10801	10804
607	CQUAD4	88	4	10804	10801	10802	10805
608	CQUAD4	89	4	10903	10900	10901	10904
609	CQUAD4	90	4	10904	10901	10902	10905
610	CQUAD4	91	4	11003	11000	11001	11004
611	CQUAD4	92	4	11004	11001	11002	11005
612							
613	\$						
614	\$...1... ...2... ...3... ...4... ...5... ...6... ...7... ...8... ...9... ...10...						
615	\$ MASAS PUNTUALES DEL BORDE DE ATAQUE DEL ALA						
616	CONM2	1000	10000		14.33851		
617	CONM2	1001	10003		14.33851		
618	CONM2	1002	10100		28.67702		
619	CONM2	1003	10103		28.67702		
620	CONM2	1004	10200		28.67702		
621	CONM2	1005	10203		28.67702		
622	CONM2	1006	10300		28.67702		
623	CONM2	1007	10303		28.67702		
624	CONM2	1008	10400		28.67702		
625	CONM2	1009	10403		28.67702		
626	CONM2	1010	10500		28.67702		
627	CONM2	1011	10503		28.67702		
628	CONM2	1012	10600		28.67702		
629	CONM2	1013	10603		28.67702		
630	CONM2	1014	10700		28.67702		
631	CONM2	1015	10703		28.67702		
632	CONM2	1016	10800		28.67702		
633	CONM2	1017	10803		28.67702		
634	CONM2	1018	10900		28.67702		
635	CONM2	1019	10903		28.67702		
636	CONM2	1020	11000		14.33851		
637	CONM2	1021	11003		14.33851		
638	\$ MASAS PUNTUALES DEL EJE MEDIO DEL ALA						
639	CONM2	1100	10001		28.78064		
640	CONM2	1101	10004		28.78064		
641	CONM2	1102	10101		57.56127		
642	CONM2	1103	10104		57.56127		
643	CONM2	1104	10201		57.56127		
644	CONM2	1105	10204		57.56127		
645	CONM2	1106	10301		57.56127		
646	CONM2	1107	10304		57.56127		
647	CONM2	1108	10401		57.56127		
648	CONM2	1109	10404		57.56127		



```

649 CONM2 1110 10501 57.56127
650 CONM2 1111 10504 57.56127
651 CONM2 1112 10601 57.56127
652 CONM2 1113 10604 57.56127
653 CONM2 1114 10701 57.56127
654 CONM2 1115 10704 57.56127
655 CONM2 1116 10801 57.56127
656 CONM2 1117 10804 57.56127
657 CONM2 1118 10901 57.56127
658 CONM2 1119 10904 57.56127
659 CONM2 1120 11001 28.78064
660 CONM2 1121 11004 28.78064
661 $ MASAS PUNTUALES DEL BORDE DE SALIDA DEL ALA
662 CONM2 1200 10002 38.96426
663 CONM2 1201 10005 38.96426
664 CONM2 1202 10102 77.92852
665 CONM2 1203 10105 77.92852
666 CONM2 1204 10202 77.92852
667 CONM2 1205 10205 77.92852
668 CONM2 1206 10302 77.92852
669 CONM2 1207 10305 77.92852
670 CONM2 1208 10402 77.92852
671 CONM2 1209 10405 77.92852
672 CONM2 1210 10502 77.92852
673 CONM2 1211 10505 77.92852
674 CONM2 1212 10602 77.92852
675 CONM2 1213 10605 77.92852
676 CONM2 1214 10702 77.92852
677 CONM2 1215 10705 77.92852
678 CONM2 1216 10802 77.92852
679 CONM2 1217 10805 77.92852
680 CONM2 1218 10902 77.92852
681 CONM2 1219 10905 77.92852
682 CONM2 1220 11002 38.96426
683 CONM2 1221 11005 38.96426
684
685
686 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
687 CONM2 1300 20002 -1 328.33360.0762 6.2484 0.0 +1301
688 +1301 0.0 68.25133 0.0
689
690 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
691
692 Aerodynamic Panel for Doublet-Lattice Theory
693 CAERO1 101001 100000 101 151 1 +
694 + -0.6404 0. 0.0 2.5 -0.6404 6.096 0.0 2.5
695
696 Geometry Definition
697 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
698 PAERO1 100000
699 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
700
701 List of Span Division Points
702 AEFACT 101 0. 0.05 0.1 0.15 0.2 0.25 0.3 +
703 + 0.35 0.4 0.45 0.5 0.54 0.58 0.62 0.66 +
704 + 0.7 0.72 0.74 0.76 0.78 0.8 0.82 0.84 +
705 + 0.86 0.88 0.9 0.92 0.94 0.96 0.98 1.0
706 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
707
708 List of Chord Division Points
709 AEFACT 151 0. 0.02 0.04 0.06 0.08 0.1 0.12 +
710 + 0.14 0.16 0.18 0.2 0.22 0.24 0.26 0.28 +
711 + 0.3 0.32 0.34 0.36 0.38 0.4 0.45 0.5 +
712 + 0.54 0.58 0.62 0.66 0.7 0.72 0.74 0.76 +
713 + 0.78 0.8 0.82 0.84 0.86 0.88 0.9 0.92 +
714 + 0.94 0.96 0.98 1.0
715 $. . . 1 . . . | . . . 2 . . . | . . . 3 . . . | . . . 4 . . . | . . . 5 . . . | . . . 6 . . . | . . . 7 . . . | . . . 8 . . . | . . . 9 . . . | . . . 10 . . .
716
717
718 Surface Spline for Interpolating Forces
719 SPLINE1 150000 101001 101001 102260 111111
720

```

```

721
722 SET1 111111 10000 10001 10002 10100 10101 10102 10200 +
723 + 10201 10202 10300 10301 10302 10400 10401 10402 +
724 + 10500 10501 10502 10600 10601 10602 10700 10701 +
725 + 10702 10800 10801 10802 10900 10901 10902 11000 +
726 + 11001 11002
727 List of structural grid points for the spline
728
729 $
730 $ Modal Analysis
731 $
732 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
733
734 Real Eigenvalue Extraction Method: Lanczos
735 EIGRL 200 0.0 50. MASS
736 $ Frequency range of interest Normalize to unit value of the generalized mass
737
738
739 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
740 $ VELOCITY REFC RHOREF SIMXZ
741 AERO 100. 2.5 1.0 +1 0
742 Basic parameters for unsteady aerodynamics
743
744 $
745 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
746 Flutter Data METHOD DENS MACH VEL IMETH Number of modes
747 FLUTTER 300 PKNL 100 200 300 6
748 PK Method Pointers to FLFACTs
749 No Looping
750
751 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
752 Mach Number - Frequency Table for Aerodynamic Matrix Calculation
753
754 MKAERO1 0.8 +
755 + 0.001 0.005 0.01 0.02 0.03 0.04 0.05 0.06 +
756 MKAERO1 0.8 +
757 + 0.07 0.08 0.1 0.2 0.3 0.5 0.75 1. +
758 $ MACH
759 $...1...|...2...|...3...|...4...|...5...|...6...|...7...|...8...|...9...|...10...
760
761 Mach Numbers for Flutter Analysis
762 FLFACT 200 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
763 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
764 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
765 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
766 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
767 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
768 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
769 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
770 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
771 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
772 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
773 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
774 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
775 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
776 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
777 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
778 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
779 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
780 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
781 + 0.800 0.800 0.800 0.800 0.800 0.800 0.800 0.800 +
782 + 0.800 0.800
783 $ DENSITY
784
785 Density Ratios for Flutter Analysis
786 FLFACT 100 0.0132 0.0136 0.0141 0.0145 0.0150 0.0154 0.0159 +
787 + 0.0164 0.0169 0.0175 0.0180 0.0186 0.0192 0.0198 0.0204 +
788 + 0.0210 0.0217 0.0224 0.0231 0.0238 0.0246 0.0254 0.0262 +
789 + 0.0270 0.0279 0.0288 0.0297 0.0306 0.0316 0.0326 0.0337 +
790 + 0.0348 0.0359 0.0370 0.0382 0.0395 0.0407 0.0420 0.0434 +
791 + 0.0448 0.0463 0.0478 0.0493 0.0509 0.0526 0.0543 0.0560 +
792 + 0.0579 0.0597 0.0617 0.0637 0.0658 0.0679 0.0702 0.0725 +

```

```

793 + 0.0748 0.0773 0.0798 0.0825 0.0852 0.0880 0.0909 0.0938 +
794 + 0.0968 0.0999 0.1031 0.1064 0.1098 0.1133 0.1170 0.1207 +
795 + 0.1246 0.1286 0.1327 0.1369 0.1413 0.1458 0.1505 0.1553 +
796 + 0.1603 0.1655 0.1708 0.1762 0.1819 0.1877 0.1937 0.1999 +
797 + 0.2063 0.2129 0.2197 0.2268 0.2340 0.2415 0.2493 0.2573 +
798 + 0.2655 0.2740 0.2828 0.2918 0.3012 0.3108 0.3208 0.3311 +
799 + 0.3417 0.3526 0.3639 0.3735 0.3830 0.3928 0.4027 0.4129 +
800 + 0.4232 0.4337 0.4445 0.4554 0.4665 0.4779 0.4894 0.5012 +
801 + 0.5131 0.5253 0.5378 0.5504 0.5633 0.5763 0.5897 0.6032 +
802 + 0.6170 0.6311 0.6453 0.6599 0.6746 0.6897 0.7049 0.7205 +
803 + 0.7363 0.7523 0.7686 0.7852 0.8021 0.8193 0.8367 0.8544 +
804 + 0.8724 0.8907 0.9092 0.9281 0.9473 0.9667 0.9865 1.0066 +
805 + 1.0270 1.0477 1.0687 1.0900 1.1117 1.1337 1.1560 1.1787 +
806 + 1.2017 1.2250
807 $
808 $VELOCITY
809
810 Velocities for Flutter Analysis
811 FLFACT 300 242.54 242.44 242.33 242.22 242.12 242.01 241.91 +
812 + 241.80 241.69 241.59 241.48 241.37 241.27 241.16 241.05 +
813 + 240.95 240.84 240.73 240.63 240.52 240.41 240.31 240.20 +
814 + 240.09 239.98 239.88 239.77 239.66 239.55 239.45 239.34 +
815 + 239.23 239.12 239.02 238.91 238.80 238.69 238.59 238.48 +
816 + 238.37 238.26 238.15 238.05 237.94 237.83 237.72 237.61 +
817 + 237.51 237.40 237.29 237.18 237.07 236.96 236.85 236.75 +
818 + 236.64 236.53 236.42 236.31 236.20 236.09 236.09 236.09 +
819 + 236.09 236.09 236.09 236.09 236.09 236.09 236.09 236.09 +
820 + 236.09 236.09 236.09 236.09 236.09 236.09 236.09 236.09 +
821 + 236.09 236.09 236.09 236.09 236.09 236.09 236.09 236.09 +
822 + 236.09 236.09 236.09 236.09 236.09 236.09 236.09 236.09 +
823 + 236.09 236.09 236.09 236.09 236.09 236.09 236.09 236.09 +
824 + 236.09 236.09 236.09 236.80 237.51 238.21 238.91 239.61 +
825 + 240.31 241.00 241.69 242.38 243.07 243.76 244.44 245.13 +
826 + 245.81 246.49 247.17 247.84 248.52 249.19 249.86 250.53 +
827 + 251.19 251.86 252.52 253.18 253.84 254.50 255.16 255.81 +
828 + 256.46 257.11 257.76 258.41 259.06 259.70 260.35 260.99 +
829 + 261.63 262.27 262.90 263.54 264.17 264.80 265.44 266.07 +
830 + 266.69 267.32 267.94 268.57 269.19 269.81 270.43 271.05 +
831 + 271.66 272.28
832 $
833 ENDDATA
834
835
836
837
838
839
840
841
842
843

```

## 9.2 Aerodynamic mesh for flutter analysis

In order to carry out the flutter analysis of the wing, the Doublet-Lattice Method is used to calculate the aerodynamic unsteady forces. This method requires the wing to be divided into panel elements. The aerodynamic mesh of the model is show in Figure 9.1. The mesh has 10 sections along the chord following a cosinus distribution law and sixteen sections along the span following an exponential distribution law.

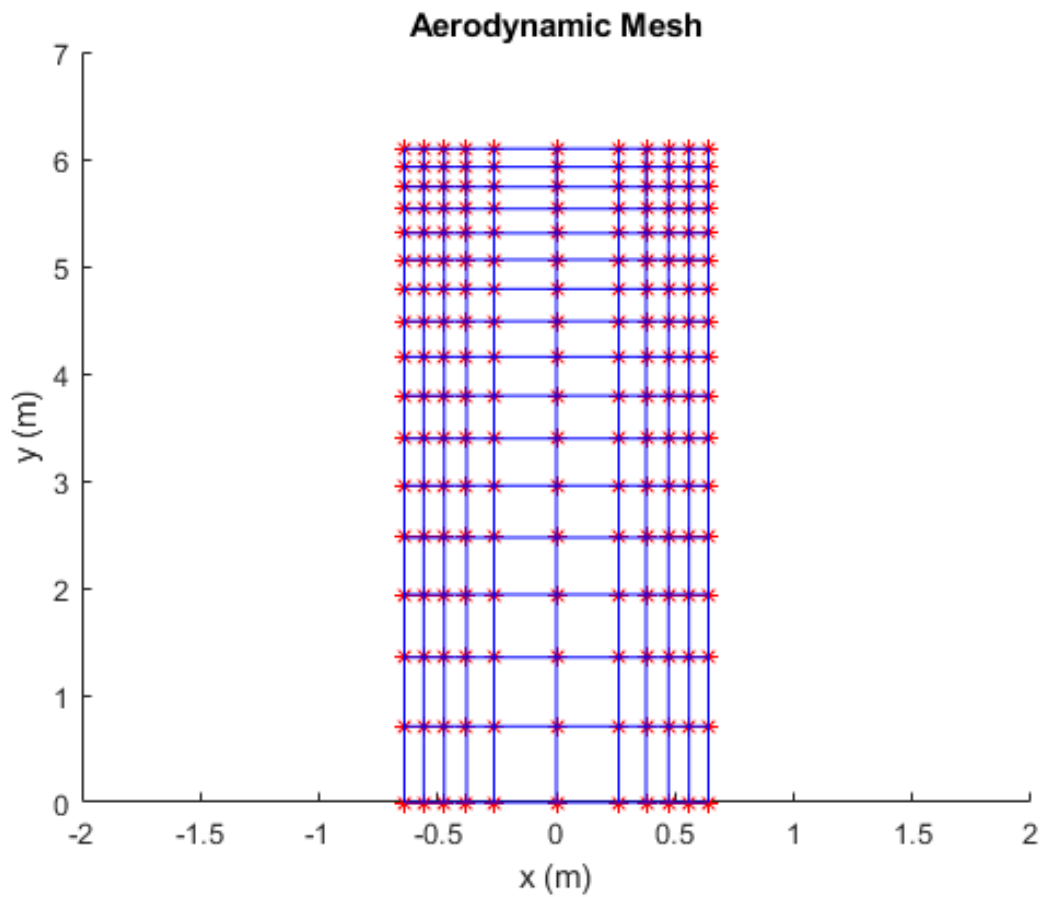


Figure 9.1: Aerodynamic Mesh.

## 9.3 Matlab-Nastran Interface

### 9.3.1 Description

An interface has been developed along this project in order to link Matlab and Nastran. This interface allows users to do a modal parametric analysis and a flutter parametric analysis in a user-friendly environment. Matlab is the software in charge of the parametric analysis and Nastran is in charge of the structural-dynamics and aeroelastic simulations.

The interface has been programmed using Matlab and it has the following functions:

1. Create .bdf input file for a SOL103 analysis in Nastran of the wing box model for different values of  $k_v$  and  $k_\theta$ .
2. Create .bdf input file for a SOL145 analysis in Nastran of the wing box model for different values of  $k_v$  and  $k_\theta$ , aerodynamic mesh and flight conditions( $M_\infty, h$ ).
3. Execute Nastran by loading a .bdf input file.
4. Read .f06 results file from a SOL103 analysis in Nastran and get natural frequencies and displacements of desired nodes of the model.
5. Read .f06 results file from a SOL145 analysis in Nastran, plot V-g and V-f diagrams, get the static divergence speed and get the flutter speed of the model.

### 9.3.2 Interface Schematic

The interface can be described by the following schematic:

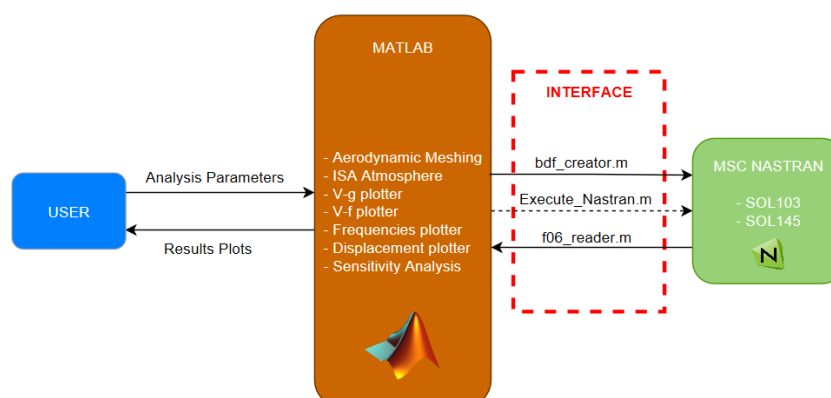


Figure 9.2: Interface Functional Schematic

## 9.4 Matlab code used to obtain modal response analysis for a range of $k_v$ and $k_\theta$

```

1 function calculate_dynamic_response_Main
2
3 k_f = logspace(0,7,60);
4 k_th=logspace(0,7,60);
5
6 freq_all_modes = zeros(4,length(k_f),length(k_th));
7 v20001_allmodes = zeros(4,length(k_f),length(k_th)); %Vertical
   displacements node 2001
8 v20005_allmodes = zeros(4,length(k_f),length(k_th)); %Vertical
   displacements node 2005
9 th20001_allmodes = zeros(4,length(k_f),length(k_th)); %Twist node
   20001
10 th20005_allmodes = zeros(4,length(k_f),length(k_th)); %Twist node
   20005
11 ratio_v_mode1 = zeros(length(k_f),length(k_th));
12 ratio_v_mode2 = zeros(length(k_f),length(k_th));
13 ratio_th_mode1 = zeros(length(k_f),length(k_th));
14 ratio_th_mode2 = zeros(length(k_f),length(k_th));
15
16 for j=1:length(k_th)
17     for i=1:length(k_f)
18         %Change the value of stiffnesses k_f and k_th in the bdf
           file
19         change_bdf(k_f(i),k_th(j));
20         %Run Nastran with the new bdf file
21         run_Nastran
22         %Read f06 file to obtain modes frequencies and
           eigenvector values
23         %in node 20001(pod) and node 20005 (wing)
24         [freq,u20001,u20005] = f06_reader('
           modos_propios_muelles_sinPunch_editado.f06');
25         freq_all_modes(:,i,j) = freq;
26         v20001_allmodes(:,i,j) = u20001(:,1);
27         v20005_allmodes(:,i,j) = u20005(:,1);
28         th20001_allmodes(:,i,j) = u20001(:,2);
29         th20005_allmodes(:,i,j) = u20005(:,2);
30     end

```

```

31 end
32
33 v_20001_model1 = zeros(length(k_f),length(k_th));
34 v_20001_model1(:, :) = v20001_allmodes(1, :, :);
35 v_20005_model1 = zeros(length(k_f),length(k_th));
36 v_20005_model1(:, :) = v20005_allmodes(1, :, :);
37 freq_model1(:, :) = zeros(length(k_f),length(k_th));
38 freq_model1(:, :) = freq_all_modes(1, :, :);
39 freq_mode2(:, :) = zeros(length(k_f),length(k_th));
40 freq_mode2(:, :) = freq_all_modes(2, :, :);
41 ratio_v_model1(:, :) = v20001_allmodes(1, :, :)./v20005_allmodes
    (1, :, :);
42 ratio_v_model2(:, :) = v20001_allmodes(2, :, :)./v20005_allmodes
    (2, :, :);
43 ratio_th_model1(:, :) = th20001_allmodes(1, :, :)./th20005_allmodes
    (1, :, :);
44 ratio_th_model2(:, :) = th20001_allmodes(2, :, :)./th20005_allmodes
    (2, :, :);
45
46 figure
47 mesh(k_f, k_th, ratio_v_model1),
48 set(gca, 'XScale', 'log');
49 set(gca, 'YScale', 'log');
50 set(gca, 'ZScale', 'log');
51 title('First mode vertical displacements response')
52 xlabel('k_{\theta}(Nm/rad)'), ylabel('k_v(N/m)'), zlabel('v_{pod}/
    v_{wing}'),
53
54 figure
55 mesh(k_f, k_th, ratio_th_model2),
56 set(gca, 'XScale', 'log');
57 set(gca, 'YScale', 'log');
58 set(gca, 'ZScale', 'log');
59 title('Second mode torsional response')
60 xlabel('k_{\theta}(Nm/rad)'), ylabel('k_v(N/m)'), zlabel('\theta_{
    pod}/\theta_{wing}'),
61
62 figure
63 mesh(k_f, k_th, freq_model1),
64 set(gca, 'XScale', 'log');

```

```

65 set(gca, 'YScale', 'log');
66 title('First mode frequency')
67 xlabel('k_{\theta}(Nm/rad)'), ylabel('k_v(N/m)'), zlabel('First
    mode frequency $\displaystyle(\frac{\text{rad}}{\text{s}})$', 'interpreter', '
    latex'),
68
69 figure
70 mesh(k_f, k_th, freq_mode2),
71 set(gca, 'XScale', 'log');
72 set(gca, 'YScale', 'log');
73 title('Second mode frequency')
74 xlabel('k_{\theta}(Nm/rad)'), ylabel('k_v(N/m)'), zlabel('Second
    mode frequency $\displaystyle(\frac{\text{rad}}{\text{s}})$', 'interpreter', '
    latex'),
75 end
76
77 function change_bdf(k_f, k_th) %Changes value of stiffnesses k_f
    and k_th in the bdf file
78 fileID = fopen('modos_propios_muelles_sinPunch_parte1.bdf', 'r');
79 A=fscanf(fileID, '%c');
80 fclose(fileID);
81 fileID = fopen('modos_propios_muelles_sinPunch_parte2.bdf', 'r');
82 B=fscanf(fileID, '%c');
83 fclose(fileID);
84 fileID = fopen('modos_propios_muelles_sinPunch_editado.bdf', 'w');
85 k_f_string=num2str(k_f, '%.2E');
86 k_th_string=num2str(k_th, '%.2E');
87 s_fl=['CELAS2  2105      ', k_f_string, '20005  3      20001  3 '
    ];
88 s_tor=['CELAS2  2106      ', k_th_string, '20005  5      20001  5
    '];
89
90 fprintf(fileID, '%s\n', A);
91 fprintf(fileID, '%s\n', s_fl);
92 fprintf(fileID, '%s\n', s_tor);
93 fprintf(fileID, '%s\n', B);
94 fclose(fileID);
95 end
96
97 function run_Nastran

```



```

98 %Delete previous Nastran results file and order nastran to run a
99 %new analysis with the new bdf file
100 delete('C:\Users\Alfonso\Documents\universidad\master\2
    _Cuatrimestre\Aeroelasticidad Avanzada\Trabajo\Apartado4\
    modos_propios_muelles_sinPunch_editado.f04');
101 delete('C:\Users\Alfonso\Documents\universidad\master\2
    _Cuatrimestre\Aeroelasticidad Avanzada\Trabajo\Apartado4\
    modos_propios_muelles_sinPunch_editado.f06');
102 delete('C:\Users\Alfonso\Documents\universidad\master\2
    _Cuatrimestre\Aeroelasticidad Avanzada\Trabajo\Apartado4\
    modos_propios_muelles_sinPunch_editado.log');
103 %Run Nastran
104 status = system('C:\MSC.Software\
    MSC_Nastran_and_Patran_Student_Editions\20190\Nastran\bin\
    nastranw.exe modos_propios_muelles_sinPunch_editado.bdf');
105 %Gives time to nastran to generate the result files
106 pause(6)
107 end
108
109 function [frec ,u20001 ,u20005] = f06_reader(filename)
110 A = fileread(filename);
111 u20005_index = strfind(A, '20005      G');
112 u20001_index = strfind(A, '20001      G');
113 u20005_matrix = zeros(4,6);
114 u20001_matrix = zeros(4,6);
115 for i=1:4
116     u20005_matrix(i,:) = str2num(A(u20005_index(i)+17:
        u20005_index(i)+105));
117     u20001_matrix(i,:) = str2num(A(u20001_index(i)+17:
        u20001_index(i)+105));
118 end
119 frequency_index = strfind(A, 'STIFFNESS');
120 frec_matrix = str2num(A((frequency_index(1)+10):(frequency_index
    (1)+10+121*4)));
121 frec = frec_matrix(:,4);
122 u20001 = u20001_matrix(:,[3,5]);
123 u20005 = u20005_matrix(:,[3,5]);
124 end

```



# Bibliography

- [1] FRAHM, H. Device for damping vibrations of bodies.
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